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Mixed H_2/H_∞ Pitch Control of Wind Turbine with a Markovian Jump Model

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Abstract- This paper proposes a Markovian jump model and the corresponding H_2/H_∞ control strategy for the wind turbine driven by the stochastic switching wind speed, which can be used to regulate the generator speed in order to harvest the rated power while reducing the fatigue loads on the mechanical side of wind turbine. Through sampling the low-frequency wind speed data into separate intervals, the stochastic characteristic of the steady wind speed can be represented as a Markov process, while the high-frequency wind speed in the each interval is regarded as the disturbance input. Then, the traditional operating points of wind turbine can be divided into separate subregions correspondingly, where the model parameters and the control mode can be fixed in each mode. Then, the mixed H_2/H_∞ control problem is discussed for such a class of Markovian jump wind turbine working above the rated wind speed to guarantee both the disturbance rejection and the mechanical loads objectives, which can reduce the power volatility and the generator torque fluctuation of the whole transmission mechanism efficiently. Simulation results for a 2 MW wind turbine show the effectiveness of the proposed method.

Keywords: Wind turbine, Markovian jump systems, mixed H_2/H_∞ control, pitch control.

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1 Introduction

In recent years, the increasing size of wind turbine has been inducing heavier subsystems and higher mechanical stresses on the turbine, while more installed capacity of wind turbines causes a larger fraction of wind power in the power grid. Accordingly, the wind turbine technology has been advancing rapidly while new challenges are appearing for the future growth of the technology. Many experts and scholars have put into the study of seeking some better control strategies to overcome these potential challenges. Currently, the acknowledged strategy of wind turbine is focusing on both power and fatigue loads, that is, to seek the maximum wind energy conversion efficient when below the rated wind speed, or to stabilize the output power to the rated power when above the rated wind speed, which is in the premise of less fatigue loads. Especially, when the wind turbine works above the rated wind speed, the pitch control is used to reduce the overload phenomenon on the mechanical and electrical parts of the unit, see [1, 2, 3, 4]. Also, the volatility of the wind speed can lead to the large range variety of operating points, while the turbulent wind speed can lead to the extra fatigue loads and output power ripple, which will cause a significant negative impact on the wind turbine mechanical side and the stability of the power grid [5]. Therefore, the control objective is to change the wind energy utilization coefficient of wind turbine through using the pitch controller, to stabilize the output power near the power rating adapting to the large range variety of operating points, restrain the wind disturbance and reduce the fatigue loads [6].

Traditional techniques concentrate on the control design based on several operating points due to the strong nonlinearity of wind turbine, and the proportional-integral (PI) control is usually adopted for one or more operating points. However, when the operating point deviates from the operating point, the corresponding control effect will decline. Moreover, the stochastic characteristic of wind speed causes the the frequent switchings of wind turbine operating points, which brings further difficulty for control design to satisfy the above mentioned control strategy. Many authors have widely applied the modern control theory in the design of wind turbine control, such as linear-parameter-varying (LPV) control, model predictive control (MPC) or nonlinear feedback control, see [7, 8, 9, 10, 11, 12, 13, 18, 19, 33, 21, 22, 23, 28, 29, 30, 33]. Especially, [7] and [8] have designed the control law for wind turbine based on the gain scheduling method, where the switching law is satisfying a specific condition. However, it is very complicated to solve the LPV controller [11], and the switching is far away from the realistic stochastic property.

On the other hand, there have been many research related to the stochastic property of wind speed in a specified wind farm, see [14, 15, 16, 17], etc. Especially, [14] and [15] have analyzed the time series data of wind speed by applying the Markov process. Most work only concentrate on the static information of wind speed for choosing farm, or generate wind speed for testing. Meanwhile, Markovian jump system

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has been well investigated due to the probabilistic description of model parameters switchings induced by external causes, e.g., random faults, unexpected events, uncontrolled configuration changes, see [26, 27] and the references therein. However, up to date, there has been no related research work combining the stochastic property of wind speed into the control strategy of wind turbine, which is an interesting topic and leads to this study. Another challenge is that, once the wind turbine has been modelled into Markovian jump systems, how to design the corresponding control to satisfy the desired performances of wind turbine, where the traditional techniques are not appropriate due to the stochastic switchings of wind turbine operating points.

This paper is aimed at regulating the generator speed of wind turbine through the state feedback H_2/H_∞ control, which is driven by the frequently switching wind speed, and the switching between operating points satisfies the stochastic property of steady wind speed. More concretely, the main contribution of this paper contains the following aspects:

- The wind turbine driven by the switching wind speed is modeled into Markovian jump systems, which has represented the stochastic characteristic of the wind speed variation into a Markov process. Through sampling the low-frequency wind speed data of the specified wind turbine into separate intervals, the stochastic characteristic of wind speed variation can be represented as a Markov process, then the traditional operating points of wind turbine can be divided into separate subregions correspondingly, where the model parameters and the control mode can be fixed in each mode.

- For the Markovian jump systems representing the wind turbine switching stochastically between different operating points, the mixed H_2/H_∞ control problem is discussed to guarantee both the disturbance rejection and the mechanical loads objectives, where the controller design constraints include H_∞ problem form better generator speed regulation, and H_2 problem for less fatigue loads.

The rest of this paper is organized as follows: Section 2 describes the Markov process model of the steady wind speed, while the high-frequency wind speed in the each interval is regarded as the disturbance input. In Section 3, due to the fact that the operating points is corresponding to the steady wind speed, the wind turbine is modeled into Markovian jump system. Then, the mixed H_2/H_∞ control problem is discussed for the linearized Markovian jump model of wind turbine in terms of LMIs in Section 4. In Section 5, the proposed method is applied on a 2MW wind turbine with the historical wind speed data. Section 6 concludes this paper.

For convenience, we adopt the following notations: A': the transpose of a matrix or vector A. $A \geq 0$ ($A > 0$): the positive semi-definite (positive-definite) matrix. I: the identity matrix. R^n: n-dimensional real Euclidean space.

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2 The Markov model for average wind speed

In this section, the stochastic property of wind speed is modelled into the Markov process for the further control design. For the wind turbine control design, several operating points are usually established corresponding to the separate wind speed due to the nonlinear terms of the wind turbine model. However, when the wind speed switches between different operating points frequently, the designed control effect will be reduced significantly. Herein, we try to extract the stochastic property of wind speed and then apply to the wind turbine control design.

Firstly, in order to extract the stochastic property of wind speed, the actual wind speed V is divided into the average wind speed V_s and disturbance wind speed V_w, which is corresponding to the low-frequency steady part and high-frequency turbulence part, respectively:

$$V(t) = V_s(t) + V_w(t). \tag{1}$$

As mentioned in [11], at any time interval t_p around t_0, the steady part V_s can be defined as

$$V_s(t) = \frac{1}{t_p} \int_{t_0 - t_p/2}^{t_0 + t_p/2} V(t)dt, \tag{2}$$

where t_p ranges from 10 to 20 minutes, and V_w corresponds to the high-frequency part whose durations are less than 10 minutes.

Motivated by the research of [14] and [15], the stochastic property of the average wind speed $V_s(t)$ can be presented as a Markov process, which is based on the transitional probability matrices of various time steps and sample datas. Most often, a first-order continuous-time Markov chain implies preservation of statistical parameters and especially the first-order autocorrelation coefficient in the synthetic sequences. In order to calculate the Markov chain transitional probabilities, initially the wind speed variation domain is divided into many states, which is determined according to the average V_s and standard deviation S_v of the available wind speed time series. The stages are arranged with the average and various standard deviations of subdivisions. The number of states is determined according to the variation domain of the wind speed values as in Table 1, which has divided the wind speed into N-regions between the rated wind speed and the cut-out wind speed.

In general, let the number of states at each time instant be N (N in Table 1 as to the above rated wind speed). Hence, there will be $N \times N$ transitions between two successive time instances. According to 6.4.2 in [18], the transition probabilities p_{ij} from a state at time k to another state at time $k+1$, *i.e.* can be represented as

$$p_{ij} = P(r_{k+1} = j \,\big|\, r_k = i),$$

which is computed by

$$p_{ij} = \frac{\textit{observed transitions from state } i \textit{ to } j}{\textit{ocurrences of state } i}. \tag{3}$$

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Accordingly, the transition probability matrix $P_{k,k+1}$ can be prepared from the observed wind speed data:

$$P_{k,k+1} = \begin{bmatrix} p_{11} & p_{12} & p_{13} & \cdots & p_{1N} \\ p_{21} & p_{22} & p_{23} & \cdots & p_{2N} \\ p_{31} & p_{32} & p_{33} & \cdots & p_{3N} \\ \vdots & \vdots & \vdots & \vdots & \vdots \\ p_{N1} & p_{N2} & p_{N3} & \cdots & p_{NN} \end{bmatrix} \tag{4}$$

with $p_{ij} \geq 0$ for $i, j \in \{1, 2, \cdots, N\}$, and

$$\sum_{j=1}^{n} p_{ij} = 1.$$

The above matrix shows all the transition probabilities p_{ij} of the average wind speed in state i to state j. Hence, through classifying the average wind speed into several regions, and sampling the wind speed data in the separate wind speed region, we can calculate the transitions probabilities (4) between different wind speed regions where the low-frequency average wind speed locates in. In this way, the stochastic properties of the wind speed can be represented into the Markov process $r(k)$:

$$V(k) = V_s(k, r(k)) + V_w(k). \tag{5}$$

3 Wind turbine with a Markovian jump model

In this section, the stochastic property of wind speed is combined with the wind turbine, which sustains the stochastic controlled wind turbine model, due to the fact that the switchings of wind turbine operating points are closely connected to the switchings of steady wind speed between different regions. The detailed modelling process is give as follows:

3.1 Wind turbine, Transmission Mechanism and Generator Subsystems

The wind turbine control system consists of the three subsystems: the wind turbine, the transmission mechanism and the generator, see Fig. 1. Following with the operating points determined by the wind speed, the wind turbine can be modelled as the two-mass model given as follows:

(i) The wind turbine is the drive of the whole system. The rotor torque can be expressed as:

$$T_r = \frac{1}{2} C_p(\lambda, \beta) \rho \pi R^2 \frac{V^3}{\omega_r}, \tag{6}$$

$$\lambda = \frac{\omega_r R}{V}, \tag{7}$$

where T_r is the rotor torque of wind turbine, R is the rotor radius, ρ is the air density, V is the wind speed, ω_r is the rotor speed, λ is the tip speed ratio, β is the pitch angle. The wind-power utilization coefficient $C_p(\lambda, \beta)$ is approximately calculated and modelled under different wind speed conditions for

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a specific wind turbine [13], which is a nonlinear function respect to λ and β, and can be expressed with the parameter c_1-c_8 as follows:

$$C_p(\lambda,\beta) = c_1\left(\frac{c_2}{\lambda^*} - c_3\beta - c_4\right)e^{-\frac{c_5}{\lambda^*}} + c_6\lambda, \tag{8}$$

with

$$\lambda^* = \left(\frac{1}{\lambda + c_7\beta} - \frac{c_8}{\beta^3+1}\right)^{-1}.$$

The dynamic characteristics of wind turbine can be expressed as:

$$J_r\dot{\omega}_r = T_r - B_{stif}\theta - K_{damp}\dot{\theta}, \tag{9}$$

where J_r is the equivalent moment of inertia for wind turbine rotor; B_{stif} is the equivalently torsional stiffness of shaft; K_{damp} is the equivalently damping factor of shaft; θ is the equivalently torsional angle of shaft, and satisfying

$$\dot{\theta} = \omega_r - \frac{1}{N_g}\omega_g.$$

The actuator of pitch angle control can be express as:

$$\dot{\beta} = \frac{1}{\tau}(\beta_r - \beta), \tag{10}$$

where β_r is the referenced pitch angle, τ is the time constant of actuator.

Assume that the drag torque T_d concentrates on the wind rotor, which can be expressed as:

$$T_d = K_d\omega_r, \tag{11}$$

where K_d is the damping coefficient of the transmission mechanism.

(ii) The dynamic characteristic of the generator can be expressed as:

$$J_g\dot{\omega}_g = \frac{B_{stif}}{N_g}\theta + \frac{K_{damp}}{N_g}\dot{\theta} - T_g, \tag{12}$$

where J_g is the equivalent moment of inertial for generator rotor; T_g is the generator torque; ω_g is the generator rotor speed. Due to the smaller time constant, it has rapid response to the order from the mechanical side. In its workspace, its characteristic can be approximated by piecewise linear functions. If ω_z is defined as the control output, the linear torque characteristic in the normal workspace can be represented as

$$T_g = B_g(\omega_g - \omega_z), \tag{13}$$

where B_g is the torque-speed curve slope of induction generators. The generator output power can be expressed as

$$P = \omega_g T_g. \tag{14}$$

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3.2 Comprehensive Markovian jump model for wind turbine

Select the state variable $x_h = [\theta, \omega_r, \omega_g, \beta]'$ and the control variable $u = [\beta_r, \omega_z]$. From (6)-(14), the nonlinear comprehensive model of wind turbine can be formulated as:

$$\dot{x}_h = f(x_h, t) + g(x_h)u, \tag{15}$$

where

$$f(x_h,t) = \begin{bmatrix} \omega_r - \frac{1}{N_g}\omega_g \\ \frac{1}{J_r}T_r - \frac{K_{damp}}{J_r}\omega_r + \frac{K_{damp}}{J_r N_g}\omega_g - \frac{B_{stif}}{J_r}\theta \\ \frac{K_{damp}}{J_g N_g}\omega_r - \left(\frac{K_{damp}}{J_g N_g^2} + \frac{B_g}{J_g}\right)\omega_g + \frac{B_{stif}}{J_g N_g}\theta \\ -\frac{1}{\tau}\beta \end{bmatrix},\ g(x_h) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & \frac{B_g}{J_g} \\ \frac{1}{\tau} & 0 \end{bmatrix}.$$

Considering (5) and (15), applying the Taylor expansion on the steady wind speed V_s, omitting the higher order terms, can be expressed as:

$$T_r = T_{r1}(V_s) + T_{r2}(V_s)V_w,$$

where

$$\begin{aligned} T_{r1}(V_s) &= T_r|_{V=V_s} = \frac{1}{2}C_P(\lambda,\beta)\rho\pi R^2\frac{V_s^3}{w_r}, \\ T_{r2}(V_s) &= \frac{\partial T_r}{\partial V}\Big|_{V=V_s} = \left(3V^2C_p(\lambda,\beta) - \frac{\partial C_p(\lambda,\beta)}{\partial\lambda}\omega_r VR\right)\Big|_{V=V_s}. \end{aligned}$$

Because the steady wind speed V_s can vary the steady operating points of wind turbine, it can be defined as the varying parameter r_t to schedule the whole wind power generation process. Consequently, it yields the following nonlinear Markovian jump system model with the perturbed term for wind turbine:

$$\dot{x_h} = \widetilde{f}(x_h, t) + g_u(x_h)u + g_w(x_h, t)V_w, \tag{16}$$

where

$$\widetilde{f}(x_h,t) = \begin{bmatrix} \omega_r - \frac{1}{N_g}\omega_g \\ \frac{T_{r1}}{J_r} - \frac{K_{damp}}{J_r}\omega_r + \frac{K_{damp}}{J_r N_g}\omega_g - \frac{B_{stif}}{J_r}\theta \\ \frac{K_{damp}}{J_g N_g}\omega_r - \left(\frac{K_{damp}}{J_g N_g^2} + \frac{B_g}{J_g}\right)\omega_g + \frac{B_{stif}}{J_g N_g}\theta \\ -\frac{1}{\tau}\beta \end{bmatrix},\ g_u(x_h) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & \frac{B_g}{J_g} \\ \frac{1}{\tau} & 0 \end{bmatrix},\ g_w(x_h,t) = \begin{bmatrix} 0 \\ \frac{T_{r2}}{J_r} \\ 0 \\ 0 \end{bmatrix}.$$

For (16), we define the operating points as $\bar{x}_i = \left[\bar{\theta}_i,\, \bar{\omega}_{ri},\, \bar{\omega}_{gi}, \bar{\beta}_i\right]'$ with the parameter i which is corresponding to the current operating point, then choose the state variable

$$x = (\triangle\theta, \triangle\omega_r, \triangle\omega_g, \triangle\beta)' = (\theta - \bar{\theta}_i,\, \omega_r - \bar{\omega}_{ri},\, \omega_g - \bar{\omega}_{gi},\, \beta - \bar{\beta}_i,)'. \tag{17}$$

Note that, the operating points are chosen within the corresponding steady wind speed subregion $V_s(i)$. As shown in Table 2, the whole operating region of the wind turbine control system (16) consist of N-operating subregions.

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Then, applying the Taylor expansion around the corresponding operating points $\bar{x}_i$, system (16) can be linearized into a set of linear subsystems with the following form:

$$\dot{x} = \bar{A}(i)x + \bar{B}_1 u + \bar{B}_2(i)V_w, \tag{18}$$

where

$$\bar{A}_i = \begin{bmatrix} 0 & 1 & -\frac{1}{N_g} & 0 \\ -\frac{B_{stif}}{J_r} & \frac{1}{J_r}\frac{\partial T_{r1}}{\partial \omega_r} - \frac{K_{damp}}{J_r} & \frac{K_{damp}}{J_r N_g} & \frac{1}{J_r}\frac{\partial T_{r1}}{\partial \beta} \\ \frac{B_{stif}}{J_g N_g} & \frac{K_{damp}}{J_g N_g} & -\frac{K_{damp}}{J_g N_g^2} - \frac{B_g}{J_g} & 0 \\ 0 & 0 & 0 & -\frac{1}{\tau} \end{bmatrix}, \ \bar{B}_1 = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & \frac{B_g}{J_g} \\ \frac{1}{\tau} & 0 \end{bmatrix}, \ \bar{B}_{2i} = \begin{bmatrix} 0 \\ \frac{T_{r2}}{J_r} \\ 0 \\ 0 \end{bmatrix}.$$

For every operating point $\bar{x}_i$, through using the zero-order holder method, the above continuous-time system (18) can be transformed into the following discrete-time system:

$$x(k+1) = A(i)x(k) + B_1 u(k) + B_2(i)V_w(k). \tag{19}$$

According to the discussion in Section 2, considering the fact that the switching between the operating points satisfies the Markov process, then for all $i \in \varphi$, the above subsystems can compose the following discrete-time Markovian jump wind turbine control system:

$$x(k+1) = A(r(k))x(k) + B_1 u(k) + B_2(r(k))V_w(k). \tag{20}$$

Obviously, the above Markov process still satisfies the previous transition probability matrix (4).

Based on the above discussion, the stochastic property of wind speed has been brought into the dynamics of wind turbine in terms of Markov process, which can better describe the dynamic process of wind turbine. However, the tradition proportional-integral-derivative (PID) control or gain scheduling control is not appropriate for such a class of system. An interesting phenomenon of Markovian jump systems is that, ever all subsystem is stable with good dynamic response, the stochastic switchings can still cause the unstability, not to mention the dynamic response. Hence, there requires the corresponding control strategy to guarantee the control objectives when the operating points switching in the form of Markov process.

4 Mixed H_2/H_∞ control of wind turbine

In this section, the mixed H_2/H_∞ control problem is discussed for the linearized Markovian jump model of wind turbine to guarantee both the disturbance rejection and the mechanical loads objectives.

Consider the multi-objective control of the following Markovian jump model for wind turbine working up the rated wind speed:

$$\begin{cases} x(k+1) = A(r(k))x(k) + B_1 u(k) + B_2(r(k))V_w(k), \\ z_2(k) = C_1 x(k) + Du(k), \\ z_\infty(k) = C_2 x(k), \end{cases} \tag{21}$$

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where $z_2 = \triangle T_g$ represents the generator torque fluctuation of the whole transmission mechanism similar as (17), and $z_\infty = \triangle \omega_g$ represents the regulation error due to the disturbance V_w.

The detailed operating strategy of wind turbine can be introduced as follows: When the average wind speed is below the rated wind speed (from 5 to 8.5 m/s), the control is designed for the maximum capture of wind energy in the variable-speed fixed-pitch mode until the rated rotor speed, i.e., to guarantee the maximum C_P through maintaining the best tip speed ratio; When the rotor speed reaches the rated rotor speed following with the increasing wind speed (from 8.5 to 12 m/s), the wind turbine is operating in fixed-speed fixed-pitch mode until the rated power; When above the rated wind speed (from 12 m/s to the cut-off wind speed), the control is designed to change the wind energy utilization coefficient of wind turbine through using the pitch controller and regulate the generator speed in order to harvest the rated electrical power, which is running in the variable-pitch fixed-speed mode, see [6]. Obviously, all the state reference values and control modes of wind turbine can be determined by the average wind speed.

However, the traditional control is designed separately according to the operating regions. But the stochastic wind speed causes the frequent switchings between different operating points and regions. Hence, we adopt the mixed H_2/H_∞ control problem for system (21) to guarantee both the disturbance rejection and the mechanical loads objectives. More concretely, the controller design constraints include the H_∞ problem for better speed regulation, and H_2 problem for optimizing control action to reduce the the generator torque fluctuation of the whole transmission mechanism.

To prove that the controlled system guarantees the disturbance rejection of level γ, let us consider the following cost function:

$$J_\infty = \frac{E\left[\sum\limits_{k=0}^{\infty} z'_\infty(k) z_\infty(k)\right]}{E\left[\sum\limits_{k=0}^{\infty} V'_w(k) V_w(k)\right]} \leq \gamma^2.$$

The H_∞ performance requires that under the zero initial conditions, the systems satisfies that $J_\infty \leq 0$. First of all, we take the following Lyapunov candidate

$$V(x(k), r(k)) = x'(k) P(r_k) x(k), \quad r_k = i \in \varphi,$$

then for system (21) under the control law $u(k) = K(r(k))x(k)$, we can obtain that

$$E\Big[\big(V(x(k+1), r(k+1)|x(k), r(k) = i\big) - V(x(k), i)\Big] = [x'(k), V'_w(k)]\Pi(i) \begin{bmatrix} x(k) \\ V_w(k) \end{bmatrix},$$

where

$$\Pi(i) = \begin{bmatrix} (A_i + B_{1i}K_i)' \sum\limits_{j=1}^{N} p_{ij} P_j (A_i + B_{1i}K_i) - P_i & \cdots & (A_i + B_{1i}K_i)' \sum\limits_{j=1}^{N} p_{ij} P_j B_2(i) \\ B'_{2i} \sum\limits_{j=1}^{N} p_{ij} P_j (A_i + B_{1i}K_i) & & B'_{2i} \sum\limits_{j=1}^{N} p_{ij} P_j B_{2i} \end{bmatrix}.$$

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Combining the definition of J_∞, it is easy to obtain the following:

$$\begin{aligned} & E\Big[\big(V(x(T),r(T))|x_0,r_0\big)-V(x_0,r_0)\Big] \\ = & \sum_{k=0}^{T} E\Big[\big(V(x(k+1),r(k+1)|x(k),r(k)\big)-V(x(k),i)\Big] \\ = & \sum_{k=0}^{T}[x'(k),V_w'(k)]\Gamma(i)\left[\begin{array}{c} x(k) \\ V_w(k) \end{array}\right]-\left(\sum_{k=0}^{T}(z'(k)z(k)-\gamma^2V_w'(k)V_w(k))\right), \end{aligned}$$

where

$$\Gamma(i)=\left[\begin{array}{cc} (A_i+B_{1i}K_i)'\sum\limits_{j=1}^{N}p_{ij}P_j(A_i+B_{1i}K_i)-P_i+C_{2i}'C_{2i} & \\ B_{2i}'\sum\limits_{j=1}^{N}p_{ij}P_j(A_i+B_{1i}K_i) & \\ \cdots & \begin{array}{c}(A_i+B_{1i}K_i)'\sum\limits_{j=1}^{N}p_{ij}P_jB_{2i} \\ B_{2i}'\sum\limits_{j=1}^{N}p_{ij}P_jB_{2i}-\gamma^2I\end{array} \end{array}\right].$$

Hence, if we have $\Gamma(i)<0$, let $T\to\infty$ and considering the zero initial condition, it reduces to that

$$E\left[\sum_{k=0}^{\infty}z_\infty'(k)z_\infty(k)-\gamma^2V_w'(k)V_w(k)\right]\le -E\big(V(x(T),r(T))|x_0,r_0\big)\le 0,$$

which guarantees that the closed-loop system satisfies the H_∞ performance.

Next, we need to transform the above matrix inequality into LMI form. Let $X(i)=P^{-1}(i)$ and define $\psi(i)$ and $Y(i)$ as follows: $\psi(i)=\{\sqrt{p_{i1}}I,\ldots,\sqrt{p_{iN}}I\}$, $\phi(i)=diag\{X(1),\ldots,X(N)\}$, $Y(i)=K(i)X(i)$. Pre- and post-multiplying the previous inequality $\Gamma(i)<0$ by $\phi(i)$ and using the Schur complement lemma, we get the following result:

For a given disturbance rejection of level γ, if there exist a set of symmetric and positive-definite matrices $X=(X(1),\cdots,X(N))$ and a set of matrices $Y=(Y(1),\cdots,Y(N))$, such that

$$\left[\begin{array}{cccc} -X_i & 0 & (A_iX_i+B_1Y_i)'\psi_i & X_iC_2' \\ 0 & -\gamma^2I & B_{2i}'\psi_i & 0 \\ \psi_i'(A_iX_i+B_1Y_i) & \psi_i'B_{2i} & -\phi_i & 0 \\ C_2X_i & 0 & 0 & -I \end{array}\right]<0, \tag{22}$$

then $u(k)=K(r(k))x(k)$ is the H_∞ control for system (21). In this situation, the system with the controller is said to have a H_∞ performance γ. More specifically, the system is stochastically stable and satisfies the H_∞ performance. In other word, the wind power output of wind turbine can still track the reference input under the disturbance V_w.

On the other hand, for the mechanical load, we consider the following cost:

$$J_2=E\left[\sum_{k=0}^{\infty}z_2(k)'z_2(k)\right].$$

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Obviously, the above H_2 performance can be constrained by the trace of the cost matrix Q which satisfies the following LMIs:

$$\begin{bmatrix} Q & C_1X_i + DY_i \\ X_iC_1' + Y_i'D' & X_i \end{bmatrix} > 0. \tag{23}$$

Combining the above discussion, this control design of wind turbine is aimed at a control $u(k) = K(r(k))x(k)$ to solve the following mixed H_2/H_∞ control problem for system (21):

$$\min_u (J_\infty, J_2).$$

Obviously, the above multi-objective H_2/H_∞ control design needs to minimize J_2 and J_∞ simultaneously, which requires to seek Pareto optimal solutions to achieve the simultaneous minimization similarly as [31, 32]. Herein, we adopt the loop algorithm to choose the Pareto optimal like point. Set the H_∞ performance region $\gamma \in [\gamma_{min}, \gamma_{max}]$, then circulating solve the LMIs (22) and (23) from γ_{max} to γ_{min} with a appropriate interval, and solve the suboptimal control problem $\min Trace(Q)$, respectively. The criteria behind that choice were that the values of both J_2 and J_∞ should be as small as possible for better H_2 and H_∞ performance while a feasible solution can be obtained, which can also be adjusted following with the actual requirements from both the generator torque fluctuation and the regulation error of the generator rotor speed. If infeasible, we turn to adjust the H_∞ performance region and repeat the above procedures until the Pareto optimal like point is achieved.

Note that, the above problem can be solved through the LMI control toolbox for Matlab easily, which also implies the final mixed H_2/H_∞ state feedback controller $K(i) = Y(i)X(i)^{-1}$. Based on the above discussions, the procedure of designing the mixed H_2/H_∞ control for the Markovian jump wind turbine can be given as follows:

Step 1: Classify the average wind speed into several regions containing the operating points;

Step 2: Calculate the transition probability matrix (4) between different wind speed regions through sampling the wind speed data in the separate wind speed region;

Step 3: On the each operating point, linearize the wind turbine into the corresponding linear form (18);

Step 4: Transform the continuous-time system into discrete-time system, and combine the subsystems under the Markov rule;

Step 5: Choose the appropriate Pareto optimal like point, where J_2 and J_∞ reach the expected minimum values simultaneously;

Step 6: Solve the corresponding controller gain K_i for each operating point; Check whether the designed control satisfies the performance requirements. If necessary, repeat Step 5 until the control effectiveness is satisfied.

Following with the switching of the actual wind speed in separate operating regions $r(k) = i$, we only need to choose the corresponding state feedback $K(i)$, which guarantees the control performance of wind

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turbine. Note that, the advantage of this approach is that, we only need to adjust the control parameter for each operating point, then the stochastic negative effects of wind speed can be reduced without any further procedures.

5 Simulation results

In this section, the proposed method is applied on a 2MW wind turbine with the parameters shown in the following Table 3. To verify the effectiveness of proposed control method, comprehensive simulation studies are carried out based on Wind Turbine Blockset Toolbox in Matlab/Simulink platform. The Blockset is developed by Aalborg University and RISOE DTU National Laboratory, which has been used as a general developer tool for other three simulation tools: Saber, DIgSILENT and HAWC [34].

The wind-power utilization coefficient $C_p(\lambda, \beta)$ is with the following coefficients:

$$c_1 = 0.5176,\ c_2 = 116,\ c_3 = 0.4,\ c_4 = 5,\ c_5 = 21,\ c_6 = 0.0068,\ c_7 = 0.08,\ c_8 = 0.035.$$

The historical wind speed data is collected from the distributed control system (DCS) corresponding to a wind turbine installed in the northeast of China. The sampling time is $1s$, and the sampling data contain 120000 points, see Fig. 2. As follows, we present the design procedures and the technical details mentioned in Section 3.

Step 1: From Fig. 2 and by (15), we can compute the average wind speed as shown in Fig. 3. Herein, for the pitch control of wind turbine, we only need to classify the above rated wind speed into several regions. Besides, from Fig. 3, the vast majority of wind speed is under $20m/s$. Hence, we adopt the following classifying strategy as in Table 4. From 12 to $20m/s$, each wind speed subregion is fixed with $1m/s$ range, and the corresponding operating point is chosen at the middle position of each subregion, such as $12.5m/s$, etc.

Step 2: Through analysing the data of average wind speed in Fig. 3, the number of the observed transitions between the subregions can be computed as follow:

$$\begin{bmatrix} 2672 & 182 & 0 & 0 & 0 & 0 & 0 & 0 \\ 183 & 1993 & 148 & 0 & 0 & 0 & 0 & 0 \\ 0 & 148 & 1610 & 118 & 0 & 0 & 0 & 0 \\ 0 & 0 & 118 & 1038 & 72 & 0 & 0 & 0 \\ 0 & 0 & 0 & 72 & 699 & 60 & 0 & 0 \\ 0 & 0 & 0 & 0 & 60 & 349 & 35 & 0 \\ 0 & 0 & 0 & 0 & 0 & 35 & 282 & 21 \\ 0 & 0 & 0 & 0 & 0 & 0 & 21 & 136 \end{bmatrix}.$$

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Then, according to (3), we can obtain the following probability transition matrix:

$$P_{ij} = \begin{bmatrix} 0.9359 & 0.0783 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0.0641 & 0.8579 & 0.0789 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0.0637 & 0.8582 & 0.0961 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0.0629 & 0.8453 & 0.0866 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0.0586 & 0.8412 & 0.1351 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0.0722 & 0.7860 & 0.1036 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0.0788 & 0.8343 & 0.1338 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0.0621 & 0.8662 \end{bmatrix}. \quad (24)$$

Obviously, the current wind speed only shifts to the closed upper subregion, the lower one or the remaining one. Then, the Markov process $r(k)$ has been established with the probability transition matrix (24) for the average wind speed in Fig. 3. In this situation, the stochastic property of the average wind speed has been extracted by the Markov process $r(k)$.

Step 3: To obtain the wind turbine control system (18) with the coefficients in Table 5, we adopt the following linearization procedures: Firstly, we determine the operating points when the average wind speed V_s is fixed at the middle of each subregion. By (7), considering the rated rotor speed, we can calculate the tip speed ratio for each subregion. Through checking the λ-β look-up table and using the linear interpolation method, the corresponding referenced pitch angle $\bar{\beta}_i$ can be obtained, which sustains the operating points $\bar{x}_i = \left[\bar{\theta}_i(r/s), \bar{\beta}_i(^\circ), \bar{\omega}_{ri}(r/s), \bar{\omega}_{gmi}(r/s)\right]'$ as shown in Table 5. Then, the corresponding coefficients $\bar{A}_i$, $\bar{B}_{1i}$ and $\bar{B}_{2i}$ can be determined directly.

Step 4: The wind turbine control system (18) can be transformed into the discrete-time case (21) through using the zero-order holder method, which can be realized through using the Matlab software directly. Note that, the coefficients B_1 and B_{2i} can be achieved through combining the u and V_w into a common input $\bar{u}$. Similarly procedures for C_{1i} and C_{2i}.

Step 5: From the viewpoint of numerical calculation, all possible feasible solutions for the H_∞ and H_2 performances can be solved simultaneously for a given appropriate region related to γ or $Trace(Q)$. Herein, we take $\gamma \in (0, 1)$ and all possible solutions are presented in Fig. 4. Choose the Pareto-optimal-like point as $\gamma = 0.1$ and $Trace(Q) = 137.4$, which are corresponding to the H_∞ performance and H_2 performance, respectively.

Step 6: Consider the wind turbine control system (21) with the coefficients A_i, B_1, B_{2i}, C_{1i} and C_{2i}, set the disturbance rejection performance $\gamma = 0.1$, and solve the LMIs (22) and (23), which reduces to the corresponding control gain K_i and sustains the control input β_r and ω_z as in Fig. 8 and 9.

Next, to show the efficiency of the proposed control strategy, we adopt a period of actual above rated wind speed as in Fig. 5 from the point 71851s to 72940s in Fig. 2, which can be decomposed into the average wind speed V_s in Fig. 6 and disturbance wind speed V_w in Fig. 7, respectively.

Following with the switchings of average wind speed in Fig. 6, the switchings of operating points can

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be reflected by the referenced pitch angle $\bar{\beta}_i$ as shown in Fig. 8, where the control input of the proposed method and PID are also given. Through substituting the feedback control gain K_i following with the switching of operating points, the control effectiveness can be found in the Fig. 10-24.

Although the PI control is usually adopted in practice, the gain-scheduled PID control is adopted for fully comparison as shown in Fig. 10, where the operating regions have been classified as in Table 5. In order to realize the undisturbed switching control with better dynamic response, the PID control has been improved through using the linear interpolation method. More specifically, denote the current time measured wind speed as V_k, and the previous time as V_{k-1}, we adopt the following control strategy:

i) When the wind speed changes smaller and in the same operating region, set $\beta_r = \bar{\beta}_i$.

ii) When the wind speed changes larger and over operating regions, take

$$\begin{cases} \beta_r = \alpha\bar{\beta}_{i-1} + (1-\alpha)\bar{\beta}_i, \ \textit{when wind speed increase}; \\ \beta_r = (1-\alpha)\bar{\beta}_{i-1} + \alpha\bar{\beta}_i, \ \textit{when wind speed decrease} \end{cases}$$

where

$$\alpha = \frac{|V_{k-1} - V_{i-1}|}{|V_{k-1} - V_{i-1}| + |V_k - V_{i-1}|}.$$

In such a way, the gain-scheduled PID control can better handle the case of wind turbine with the high-frequency wind speed fluctuation and reduce the switching fluctuation. For each operating region, the PID control has been tuned with a high quality. However, due to the frequent switchings of operating points and the turbulent wind speed, the rotor speed and generator speed still exist many fluctuations. With the proposed method, the control effect has been improved significantly in Fig. 11-17.

Due to the proposed H_2/H_∞ optimal control, the mechanical torques including the rotor torque T_r, the drag torque T_d and the generator torque T_g have also been improved with less fluctuations in Fig. 13, 14 and 15, respectively, which are more smoothing compared with the PID control, and have shown significance improvements on the disturbance rejection. Due to these improvements especially on T_g, less fluctuations for the mechanical torques have been guaranteed compared with PID control, which can reduce the fatigue loads efficiently. Meanwhile, the mechanical power P has been regulated with less fluctuations as shown in Fig. 16 and 17.

To better quantify these improvements, the frequency analysis under the proposed method and PID control are shown in Fig. 19-22, which describes the power spectrum density (PSD) estimation of the rotor speed ω_r, the generator speed ω_g, the generator torque T_g and power P through using the music method. It can be found that, there all exist two spikes near the frequency 0.242 and 0.398, which are caused by the turbulence wind speed as shown in Fig. 18. The PID controller can reduce the two spikes to a certain extent, while the proposed controller can reduce the interference further, and the distribution of the signal power is concentrated in the low frequency band, which shows the disturbance rejection effectiveness of the proposed method.

Moreover, the frequency statistics of the generator torque and power variations under the proposed method and PID control are shown in Fig. 23 and 24. Obviously, the frequency statistics of the proposed

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method are much more concentrated on the region nearer to 0 compared with the PID control, which shows that few fluctuations have been guaranteed for both the generator torque and mechanical power simultaneously. More concretely, better frequency statistics of generator torque variations can guarantee the fatigue loads as small as possible, and better frequency statistics of power variations implies less volatility of the output power P, which can reduce the control difficulty of the DFIG to guarantee higher-quality power.

Based on the above discussion, the power quality on the mechanical side have been improved with less volatility and mechanical loads, which can reduce the reference value variation of the DFIG side within the same grid condition. Hence, through combining the stochastic wind speed into the control design of wind turbine, the designed mixed H_2/H_∞ control can provide a feasible tuning technique on the mechanical power and fatigue load performances. Moreover, this study is mainly concentrated on the case of above the rated wind speed for simplicity, which can be generalized to the full wind speed region.

6 Conclusion

This paper has proposed a Markovian jump model for the dynamic process of wind turbine driven by the switching wind speed, and discussed the mixed H_2/H_∞ control problem. Through sampling the steady wind speed data in the separate wind speed region, the system is switching from one mode to another, and the switching rule is modeled into a Markov process. Then, the corresponding mixed H_2/H_∞ control can guarantee both the disturbance rejection and the fatigue loads objectives, which has combined the switching rule into the control design of wind turbine effectively. Further efforts could be concentrated on the controller improvement for the full wind speed region to deal with the switchings between partial and full load.

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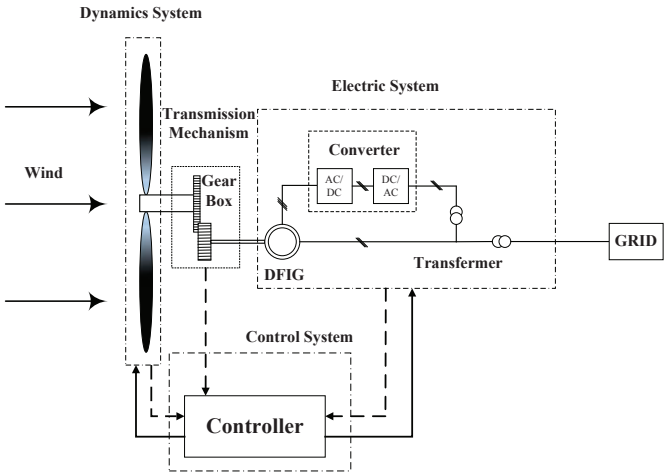

Fig. 1: The structure of wind turbine control system

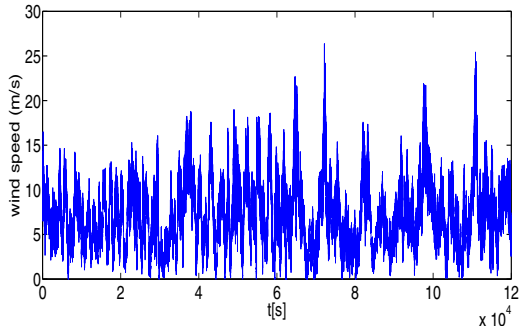

Fig. 2: The actual wind speed of wind turbine

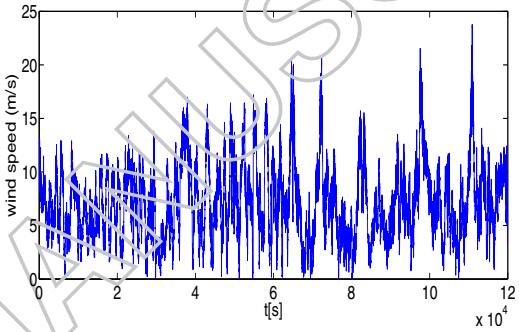

Fig. 3: The average wind speed of wind turbine

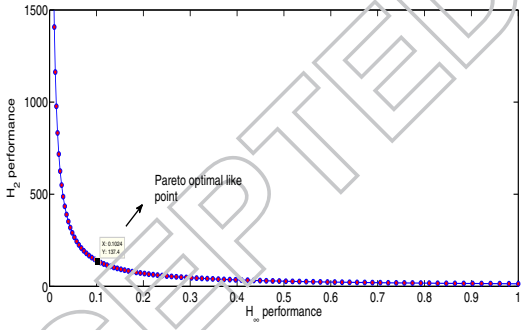

Fig. 4: The relationship of H_∞ and H_2 performance

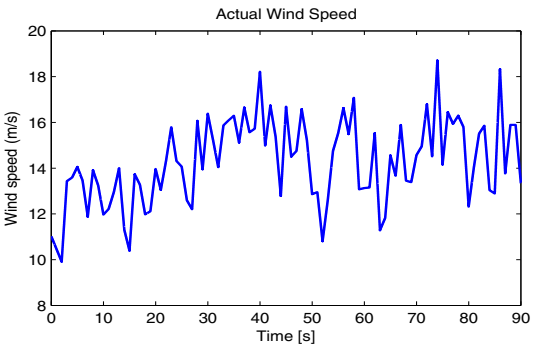

Fig. 5: The acutal wind speed

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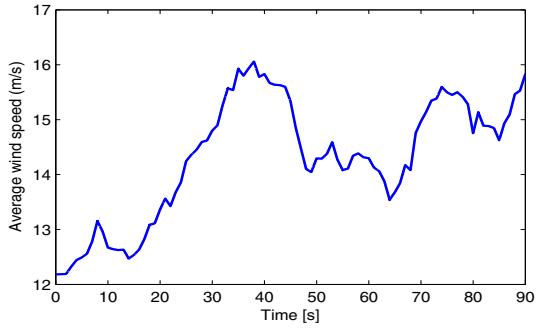

Fig. 6: The average wind speed

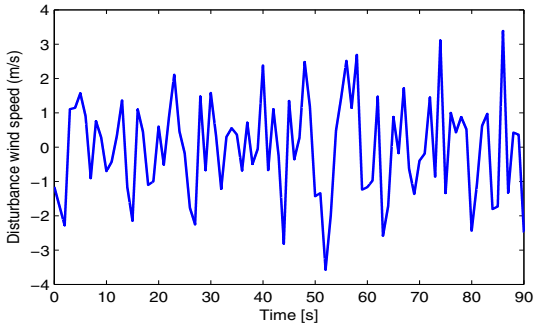

Fig. 7: The disturbance wind speed

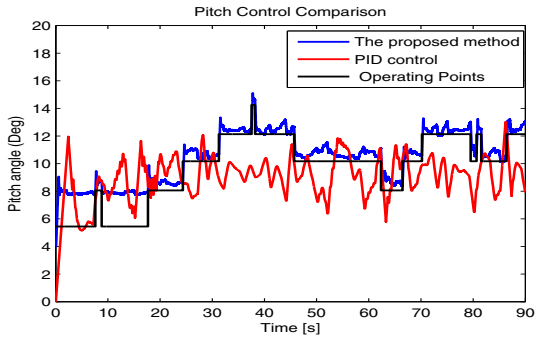

Fig. 8: Operation switchings and pitch control β_r

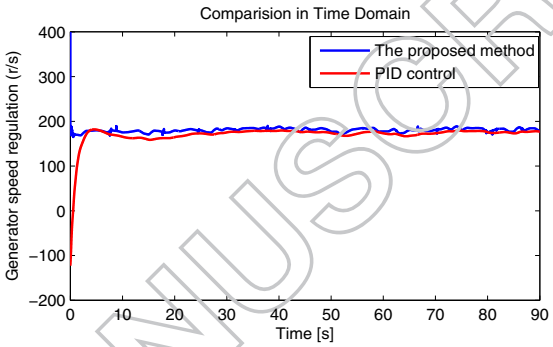

Fig. 9: The generator speed regulation ω_z

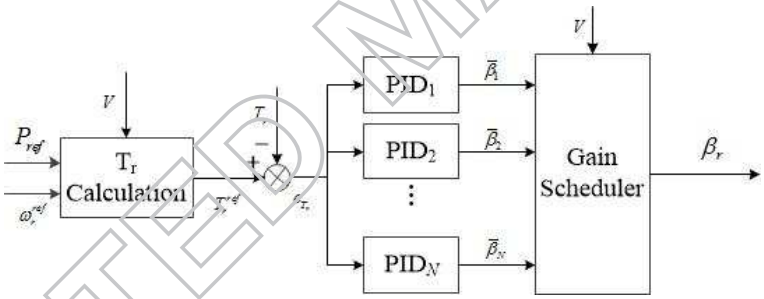

Fig. 10: The structure of wind turbine PID control system

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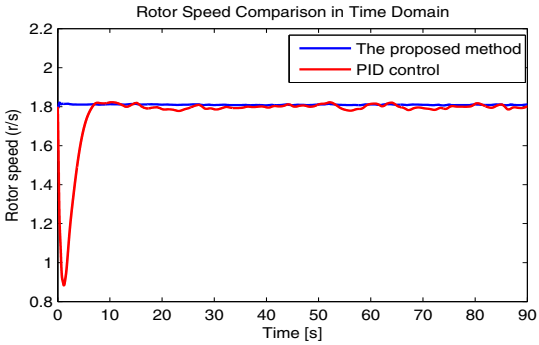

Fig. 11: The rotor speed ω_r

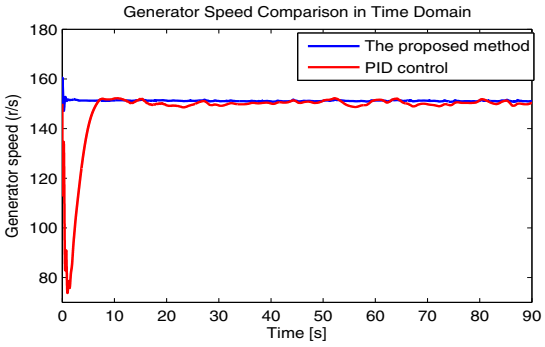

Fig. 12: The generator speed ω_g

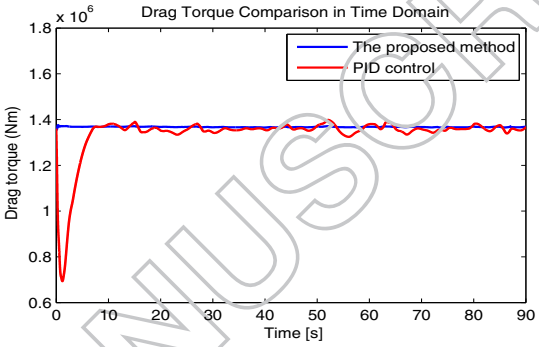

Fig. 13: The rotor torque T_r

Fig. 14: The drag torque T_d

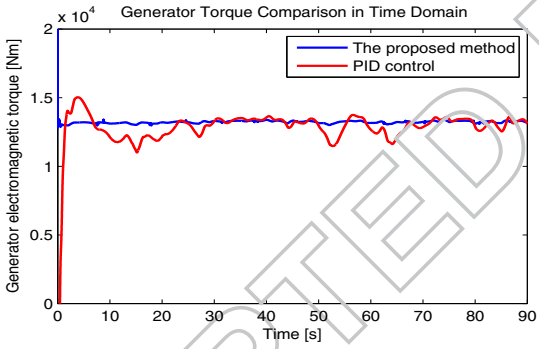

Fig. 15: The generator electromagnetic torque T_g

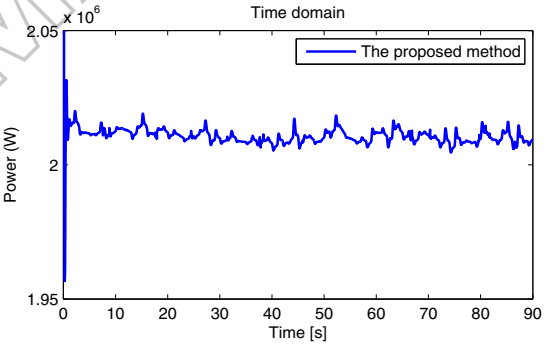

Fig. 16: The mechanical power P

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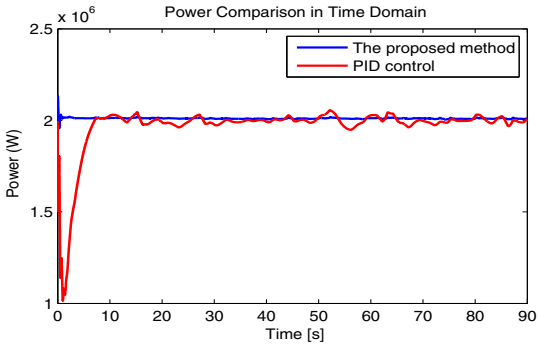

Fig. 17: Comparison of mechanical power P

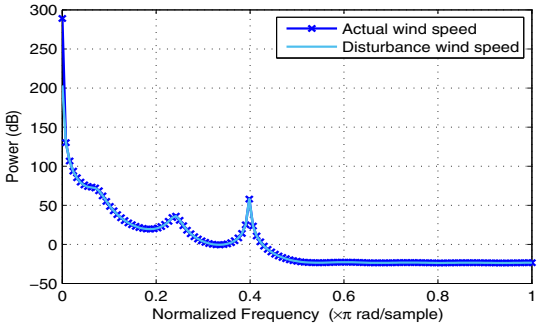

Fig. 18: PSD estimation of wind speed V and V_w

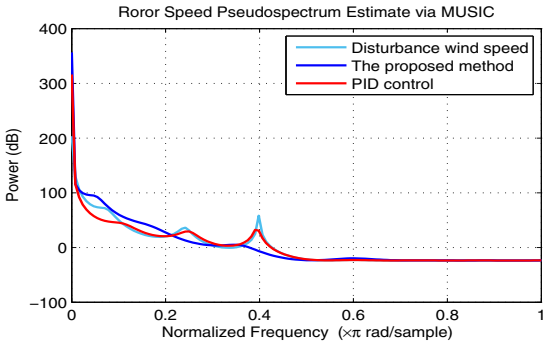

Fig. 19: PSD estimation of rotor speed ω_r

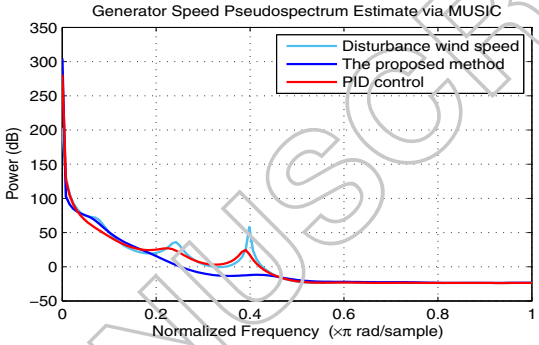

Fig. 20: PSD estimation of generator speed ω_g

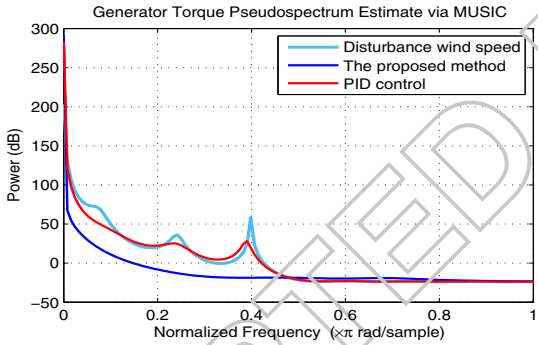

Fig. 21: PSD estimation of generator torque T_g

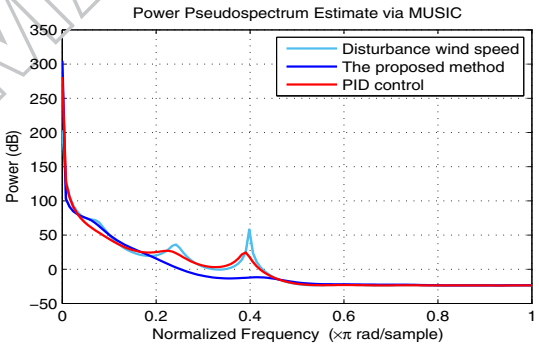

Fig. 22: PSD estimation of mechanical power P

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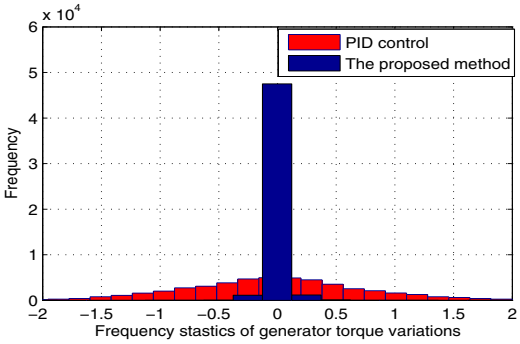

Fig. 23: Frequency statistics of T_g variations

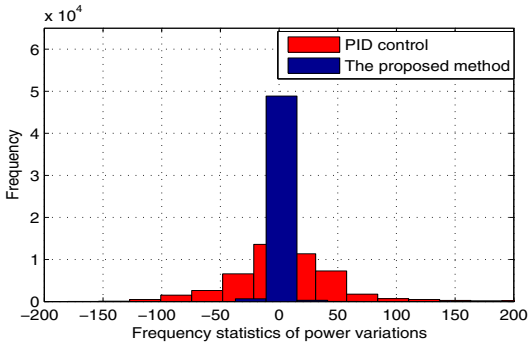

Fig. 24: Frequency statistics of power variations

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Table 1: Average Wind Speed Category

Category no.	Wind speed boundaries (m/s)
1	$V_{rated} \leq V_s \leq V_1$
2	$V_1 \leq V_s \leq V_2$
$\vdots$	$\vdots$
N	$V_{N-1} \leq V_s \leq V_{cut-out}$

Table 2: Operating Subregions Category

i	Wind speed boundaries	Operating point $\bar{x}_i$
1	$V_{rated} \leq V_s \leq V_1$	$\bar{x}_1 = \left[\bar{\theta}_1, \bar{\omega}_{r1}, \bar{\omega}_{g1}, \bar{\beta}_1\right]'$
2	$V_1 \leq V_s \leq V_2$	$\bar{x}_2 = \left[\bar{\theta}_2, \bar{\omega}_{r2}, \bar{\omega}_{g2}, \bar{\beta}_2\right]'$
$\vdots$	$\vdots$	$\vdots$
N	$V_{N-1} \leq V_s \leq V_N$	$\bar{x}_N = \left[\bar{\theta}_N, \bar{\omega}_{rN}, \bar{\omega}_{gN}, \bar{\beta}_N\right]'$

Table 3: Wind Turbine Parameters

R	$40\,m$
ρ	$1.25\,Kg/m^3$
τ	$50\,ms$
J_r	$4.95 \times 10^6\,kg \cdot m^2$
J_g	$90\,kg \cdot m^2$
N_g	83.531
B_{stif}	$1.14 \times 10^8\,N/m$
K_{damp}	$7.55658 \times 10^5\,Nm/(rad/s)$
B_g	$400\,Nm/(rad/s)$

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Table 4: Average Wind Speed Category

Category no.	Wind speed boundaries (m/s)
1	$12 \leq V_s \leq 13$
2	$13 \leq V_s \leq 14$
$\vdots$	$\vdots$
8	$19 \leq V_s \leq 20$

Table 5: Operating Subregions Category

i	$V_s(m/s)$	λ	C_p	$\bar{x}_i = \left[\bar{\theta}_i, \bar{\beta}_i, \bar{\omega}_{ri}, \bar{\omega}_{gmi}\right]'$
1	12.5	5.8240	0.3259	$\bar{x}_1 = [0,\ 5.4452,\ 1.8,\ 150.4]'$
2	13.5	5.3926	0.2587	$\bar{x}_2 = [0,\ 8.0591,\ 1.8,\ 150.4]'$
3	14.5	5.0207	0.2088	$\bar{x}_2 = [0,\ 10.1695,\ 1.8,\ 150.4]'$
4	15.5	4.6968	0.1710	$\bar{x}_4 = [0,\ 12.1346,\ 1.8,\ 150.4]'$
5	16.5	4.4121	0.1417	$\bar{x}_5 = [0,\ 14.2455,\ 1.8,\ 150.4]'$
6	17.5	4.1600	0.1188	$\bar{x}_6 = [0,\ 16.4044,\ 1.8,\ 150.4]'$
7	18.5	3.9351	0.1005	$\bar{x}_7 = [0,\ 19.2327,\ 1.8,\ 150.4]'$
8	19.5	3.7333	0.0859	$\bar{x}_8 = [0,\ 22.4339,\ 1.8,\ 150.4]'$

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