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1 **Gas-liquid two-phase flow behavior in terrain-inclined pipelines for**
2 **gathering transport system of wet natural gas**

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10 **Abstract:** The Volume of Fluid method and Re-Normalisation Group (RNG) $k-\varepsilon$
11 turbulence model were employed to predict the gas-liquid two-phase flow in a terrain-
12 inclined pipeline with deposited liquids. The simulation was carried out in a 22.5 m
13 terrain-inclined pipeline with a 150 mm internal diameter. The flow parameters were
14 numerically analyzed in detail including the phase distribution in pipes, the velocity
15 and pressure around the elbow, the liquid flow rate and liquid holdup in different cross-
16 section and the volume of liquid outflow. The numerical results presented that a wave
17 crest formed on the liquid level under the suction force which caused by the negative
18 pressure around the elbow, and then it touched to the top of the pipe. When the liquid
19 blocked the pipe, the pressure drop between the upstream and downstream of the elbow
20 increased with the increase of the gas velocity. At larger gas velocity, more liquid was
21 carried out to the pipeline. The liquid periodically flowed and returned along the uphill
22 section when the liquid was no longer flowing out of the pipeline.

23 **Keywords:** terrain-inclined pipeline; natural gas; liquid loading; gas-liquid two-phase
24 flow; Volume of Fluid

25 **1. Introduction**

26 During the development of condensate natural gas fields, such as the Puguang Gas
27 Field in China's Sichuan Basin, there are many uphill and downhill pipes due to the
28 large rolling terrain. The liquid often accumulates in the low section of the pipeline
29 during the wet natural gas transportation. The deposited liquid may block the pipeline
30 under the action of the gaseous phase and cause the sharp fluctuation of pressure [1]. It
31 affects the normal operation of the pipeline, even damages the conveying equipment.
32 In addition, the pipeline is more easily subjected to the corrosion issues [2, 3], because
33 the deposited liquid is difficult to be removed. Therefore, it is significant to study the
34 flow behavior of the deposited liquid in the wet natural gas pipelines.

35 The gas-liquid two-phase flows in the horizontal and inclined pipes have been
36 studied for decades. Taitel et al. [4] proposed a model to calculate the flow behavior
37 under the transient flow conditions in a hilly-terrain pipeline system. The model was
38 applied to the low liquid and gas flow rates, where the frictional pressure losses could
39 be neglected. Grolman and Fortuin [5] modified the apparent rough surface model to
40 predict the pressure gradient and liquid holdup in slightly inclined pipes. Salhi, et al.
41 [6] improved the one-dimensional two-fluid model to study the stability of stratified
42 gas-liquid two-phase flows in an inclined pipe. Goldstein et al. [7] proposed the exact
43 solutions for the laminar stratified flows in the inclined pipes. The solution could be
44 used for investigating the influence of the pipe inclination and flow geometry on the
45 liquid holdup and pressure gradient. Gawas et al. [8] proposed a new wave celerity
46 correlation for the gas-liquid two-phase stratified flow using the low viscosity fluids
47 and compared with a mechanistic model proposed by Watson. The Poisson probability
48 theory was employed to predict the slug frequency in the gas-liquid horizontal pipelines
49 [9]. The results indicated that the theoretical model resulted in a great improvement

50 with the average error of 15% for 1.2 m/s and 0.4 m/s superficial liquid velocities.
51 Ferrari et al. [10] developed a novel two-fluid model to capture the slug flow in pipes
52 using a one-dimensional transient hyperbolic five-equation. This theoretical model
53 successfully captured the slug onset, growing, and development from a stratified flow
54 in horizontal pipes.

55 Barnea and Shoham [11] carried out an experimental study in an air-water system
56 with 2.55 cm and 1.95 cm pipes and compared with the theoretical prediction of Taitel
57 and Dukler. The results showed that the experiment data agreed with the theoretical
58 prediction. Tzotzi and Andritsos [12] gave a modified form of Andritsos-Hanratty
59 model to study stratified gas-liquid two-phase flow and estimate pressure drop and
60 interfacial friction factor in a horizontal pipe. Jia et al. [13] used two absolute pressure
61 sensors to measure differential pressure and obtained the void fraction from differential
62 pressure model in a horizontal gas-liquid two-phase pipe. Arunkumar et al. [14] used
63 the dielectric sensors to identify the gas-liquid two-phase flow regime. The bubbly, slug
64 and stratified flow regimes could be identified in their experiments. Abdulkadir et al.
65 [15] investigated the unsteady hydrodynamic behaviour of slug flow in a horizontal
66 pipe using the an air-silicone oil mixture as the working fluid. Their results showed
67 that the drift velocity component of the bubbles in horizontal pipes far exceeds the value
68 found for vertical riser pipes. Dinaryanto et al. [16] experimentally studied the initiation
69 and flow development mechanisms of the gas-liquid two-phase slug flow in a horizontal
70 pipe., The initiation frequency of slug flow and the evolution of passing slug frequency
71 along the pipeline were also observed by using two high video cameras in their
72 experiments. Bouyahiaoui et al. [17] employed intermediary of the conductance probe
73 technique to study the slug flow behavior in the vertical downward air-water two-phase

74 pipelines. The experimental results showed that the mean averaged void fraction
75 increased with the gas superficial velocity.

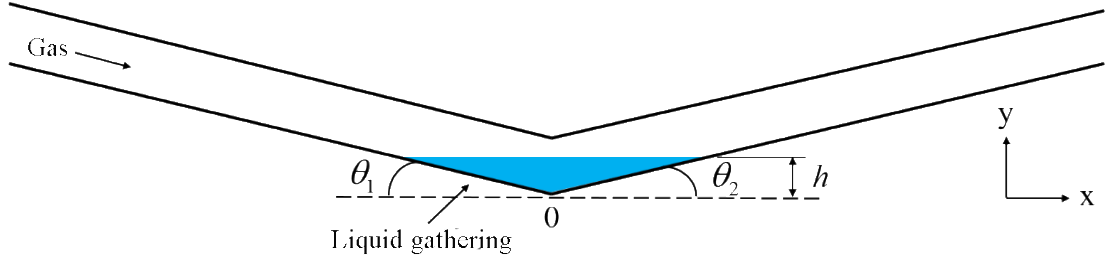
76 The numerical simulation approach was employed to predict the flow regime and
77 flow characteristics of gas-liquid two-phase flows. Mouza et al. [18] used
78 computational fluid dynamics (CFD) modeling to simulate the wave stratified two-
79 phase flow in a horizontal pipe. The distribution of the shear stresses and the velocity
80 profiles of both phases were calculated by using CFD modeling. Loilier [19] applied
81 the two-fluid model to simulate the gas-liquid two-phase flow in vertical and terrain-
82 inclined pipelines consisting of the uphill, downhill and horizontal sections. The
83 vertical bubble flow, stratified and terrain-inclined slug flows were simulated and the
84 void fraction, slug frequency, slug length and other parameters were obtained in this
85 work. Ekambara et al. [20] performed a numerical study of the bubbly two-phase flow
86 in a horizontal pipeline. The numerical results agreed well with the experimental data.
87 Vallée et al. [21] investigated the stratified and slug flow by using the CFD modeling,
88 particle image velocimetry and high-speed video observation experiments. The main
89 flow characteristics obtained by the CFD modeling were successfully validated against
90 the experiments. Verdin et al. [22] used the CFD modeling for simulating the transport
91 behavior of water droplets in 38 in. diameter pipes and compared the oil flow behavior
92 and droplets with that of water. Santim et al. [23] compared different methodologies in
93 transient isothermal gas-liquid two-phase slug flows in a horizontal pipeline. The
94 comparison studies indicated that the Drift-Flux Model presented the better agreement
95 with the pressure wave velocity by the experiments. Wang et al. [24] developed a fluid-
96 structure interaction dynamic model for the conveying severe slugging flow in a
97 pipeline-riser system. Their results showed that the dynamic response of the riser is

98 closely related to the characteristics of severe slugging, which can be used to eliminate
99 severe slugging phenomenon to improve the stability of the riser system.

100 The gas-liquid two-phase flows in horizontal and vertical pipelines attract a great
101 many of experimental and numerical studies, while the flow mechanism of the
102 gathering liquids in the elbow of hilly-terrain pipes is still not understood very well.
103 The purpose of this work is to study the gas-liquid two-phase flow in the terrain-
104 inclined pipelines to obtain local and global flow characteristics using the CFD
105 modeling, which can provide the detailed information for removing the deposited liquid
106 effectively and correspondingly protecting natural gas pipeline and equipment. The
107 Volume of Fluid (VOF) method and Re-Normalisation Group (RNG) $k-\varepsilon$ turbulence
108 model are used to predict the deposited liquid motion under the action of the gaseous
109 phase in the terrain-inclined pipeline system including an uphill section and a downhill
110 section. The flow parameters are analyzed in detail, including the velocity and pressure
111 distribution, phase fraction and cross-section liquid holdup.

112 **2. Terrain-inclined pipelines**

113 The terrain-inclined pipeline is employed for our current studies, which contains
114 a downhill section and an uphill part. The inclination angles of downhill and uphill
115 segments are both assigned to be 5° . It is assumed that a certain amount of liquids is
116 gathering at the bottom of the inclined pipeline system, as shown in Fig. 1. The three-
117 dimensional geometry of terrain-inclined pipeline is established for the computational
118 domain, including the pipe diameter of $D=150$ mm and the length of every section is
119 $75 D$, which ensures a fully developed flow for this terrain-inclined pipeline system.



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Fig. 1 Terrain-inclined pipelines

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Table 1 Flow data for gas-liquid two-phase numerical simulation

Case	U_G (m s^{-1})	ρ_G (kg m^{-3})	μ_G ($\text{m}^2 \text{s}^{-1}$)	h/D (-)	V_L (m^3)	ρ_L (kg m^{-3})	μ_L ($\text{m}^2 \text{s}^{-1}$)
1	5.5	0.6679	1.087×10^{-5}	0.75	0.01637	998.2	0.001003
2	6.5	0.6679	1.087×10^{-5}	0.75	0.01637	998.2	0.001003
3	7.5	0.6679	1.087×10^{-5}	0.75	0.01637	998.2	0.001003

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3. Computational methods

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3.1 Governing equations

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The gas-liquid two-phase flow in the terrain-inclined pipeline represents a distinct phase interface. The interface catching is a key issue for our simulation of this kind of flow behavior. The VOF model [25] uses the surface-tracking technology based on the fixed Eulerian mesh, which can be employed to model two or more immiscible fluids. Therefore, we utilize the VOF model here to track the gas-liquid phase interface in the terrain-inclined pipelines.

137 The continuity equation is as follows [26]:

$$138 \quad \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_i}(\rho u_i) = 0 \quad (1)$$

139 The momentum equation [27] is solved throughout the computational domain, and
140 the velocity field is shared among all the phases.

$$141 \quad \frac{\partial}{\partial t}(\rho \bar{u}) + \nabla \cdot (\rho \bar{u} \bar{u}) = -\nabla p + \nabla \cdot [\mu(\nabla \bar{u} + \nabla \bar{u}^T)] + \rho \bar{g} + \bar{F} \quad (2)$$

142 where ρ is the density, u is the velocity, p is the static pressure, μ is the dynamic
143 viscosity, $\rho \bar{g}$ is the gravitational body force and \bar{F} is external body force.

144 For the gas-liquid two-phase flow, the ρ and μ in each computational cell are given
145 by the following equations:

$$146 \quad \rho = \alpha_2 \rho_2 + (1 - \alpha_2) \rho_1 \quad (3)$$

$$147 \quad \mu = \alpha_2 \mu_2 + (1 - \alpha_2) \mu_1 \quad (4)$$

148 The volume fraction of each phase in each grid cell is calculated throughout the
149 domain. The interface between two phases is tracked by solving the continuity equation
150 for the volume fraction of one (or more) phases. The volume fraction equation is as
151 follows [28]:

$$152 \quad \frac{1}{\rho_q} \left[\frac{\partial}{\partial t}(\alpha_q \rho_q) + \nabla \cdot (\alpha_q \rho_q u_q) = \sum_{p=1}^n (\dot{m}_{pq} - \dot{m}_{qp}) \right] \quad (5)$$

153 where \dot{m}_{pq} is the mass transfer from phase q to phase p and \dot{m}_{qp} is the mass transfer from
154 phase p to phase q , α_q is the volume fraction of phase q .

155 The primary phase volume fraction is solved based on the following constraint:

$$156 \quad \sum_{q=1}^n \alpha_q = 1 \quad (6)$$

157 **3.2 Continuum surface force model**

158 The effect of surface force along the interface is included in the VOF model. The
159 continuum surface force model proposed by Brackbill et al. [29] is used in this paper.

160 It is implemented as a source term in the momentum equation. The pressure drop across
 161 the surface depends on the surface coefficient σ and the surface curvature are measured
 162 by two radii in orthogonal directions R_1, R_2 :

$$163 \quad p_1 - p_2 = \sigma \left(\frac{1}{R_1} + \frac{1}{R_2} \right) \quad (7)$$

164 The surface force F_{vol} is expressed as follows:

$$165 \quad F_{vol} = \sigma_{ij} \frac{\rho \kappa_i \nabla \alpha_i}{\frac{1}{2}(\rho_1 + \rho_2)} \quad (8)$$

166 3.3 Turbulence model

167 Among the k - ε , k - ω and Reynolds stress models [30-32], the RNG k - ε turbulence
 168 model [33] involves an additional term to its dissipation rate equation that can improve
 169 the accuracy for rapidly strained flow. The swirling effect on the turbulence is also
 170 included in the RNG k - ε model. Both of these two modifications show that the RNG k -
 171 ε model is more suitable for the prediction of the large curvature and strain rate flow in
 172 the terrain-inclined pipelines. Therefore, the RNG k - ε turbulence model is employed
 173 here, because the flow turns at the elbow of the pipe, which connects the uphill section
 174 and downhill section in terrain-inclined pipelines. The turbulence kinetic energy, k , and
 175 its rate of dissipation, ε , are as follows:

$$176 \quad \frac{\partial(\rho k)}{\partial t} + \frac{\partial(\rho k u_i)}{\partial x_i} = \frac{\partial}{\partial x_j} \left[\alpha_k \mu_{eff} \frac{\partial k}{\partial x_j} \right] + G_k + G_b - \rho \varepsilon - Y_M \quad (9)$$

$$177 \quad \frac{\partial(\rho \varepsilon)}{\partial t} + \frac{\partial(\rho \varepsilon u_i)}{\partial x_i} = \frac{\partial}{\partial x_j} \left[\alpha_\varepsilon \mu_{eff} \frac{\partial \varepsilon}{\partial x_j} \right] + \frac{C_{1\varepsilon} \varepsilon}{k} (G_k + C_{3\varepsilon} G_b) - C_{2\varepsilon} \rho \frac{\varepsilon^2}{k} - R_\varepsilon \quad (10)$$

178 where G_k represents the generation of turbulence kinetic energy due to the mean
 179 velocity gradients, G_b is the generation of turbulence kinetic energy due to buoyancy,
 180 Y_M represents the contribution of the fluctuating dilatation in compressible turbulence
 181 to the overall dissipation rate. α_k and α_ε are the inverse effective Prandtl numbers for k
 182 and ε , respectively.

183 **3.4 Numerical schemes**

184 The pressure-based VOF method and the RNG $k-\varepsilon$ turbulence model are employed
185 for the computation of the gas-liquid two-phase flow in the terrain-inclined pipeline.
186 The VOF explicit scheme is performed for accurately tracking the surface between the
187 gaseous and liquid phases [34]. In addition, the effect of the body force is considered,
188 including the gravity and the surface force. The pressure-velocity coupling is the PISO
189 algorithm. The continuity equation, momentum equation and turbulence equations are
190 discretized with the QUICK method, while the Geometric Reconstruction scheme
191 (Geo-Reconstruct) is chosen for the volume fraction equation.

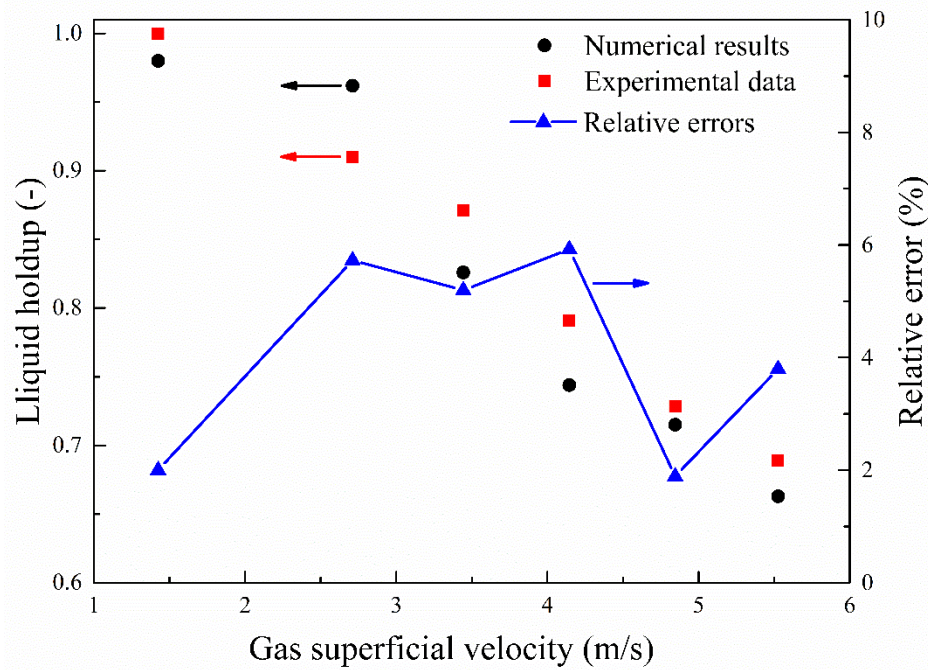
192 The structured mesh is adopted as the strategy of the grid system for the whole
193 computational domain. The velocity-inlet and outflow boundary conditions are set for
194 the inlet and outlet of the calculation model. The residual of the calculation is set to 10^{-5} ,
195 and the maximal iterations per time step is 20. The time step size is 10^{-4} s for proving
196 the calculation convergence in every time step.

197 **4. Results and discussion**

198 **4.1 CFD validation**

199 The numerical method was validated with the experiments carried out by
200 Heywood and Richardson [35] before we applied it to the gas-liquid two-phase flows
201 in the terrain-inclined pipeline. The liquid holdup of a slug flow was measured by γ -ray
202 absorption method in a horizontal pipeline with 42 mm inner diameter and 457 m length.
203 The average results of the experimental data were estimated by the probability density
204 functions. The working fluids were air and water, respectively. For the CFD model
205 validation, we utilized six group experimental data in the different gas superficial
206 velocities, while the liquid superficial velocity was fixed at 0.978 m/s. The comparison
207 of the slug flow in a horizontal pipe was described in Fig. 2. The similar flow behavior

208 of liquid holdup in the slug flow was presented between the simulations and
 209 experiments. The maximum relative error was approximate 5.9% occurring at the gas
 210 superficial velocities of 4.145 m/s. Therefore, our numerical simulation agreed well
 211 with the experimental data.



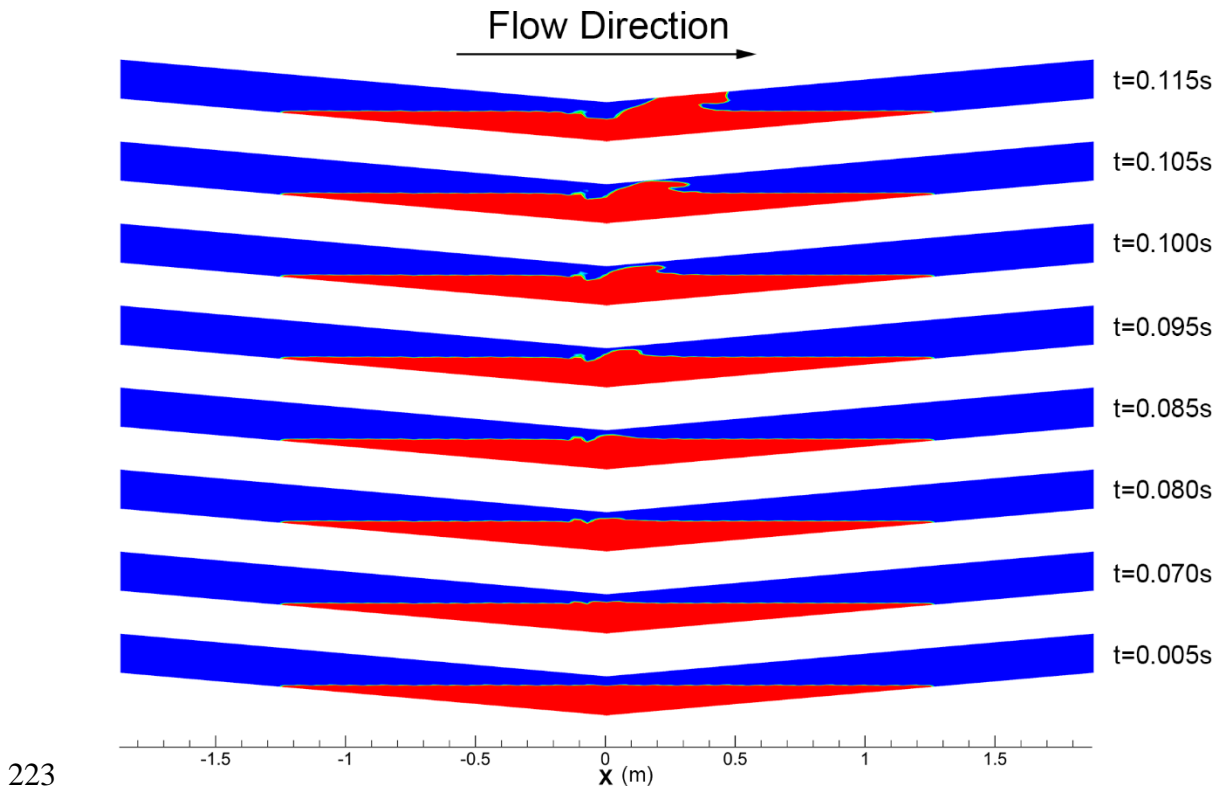
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Fig. 2. Liquid holdup in a slug flow

213

214 4.2 Process of liquid touched to the top of pipe

215 Fig. 3 shows the phase distribution near the elbow of the terrain-inclined pipe at
 216 different times with inlet gas velocity of $U_G = 5.5$ m/s. The blue contour presents the
 217 gaseous phase, while the red one reveals the liquid phase. The gas-liquid phase interface
 218 was disturbed when the gaseous phase flowed through the interface of the liquid phase.
 219 A wave crest gradually formed in the gas-liquid interface at $t = 0.005$ s – 0.085 s. Then
 220 the wave crest continued to become larger and moved along the gas flow direction at
 221 the same time ($t = 0.095$ s – 0.105 s in Fig. 3). At $t = 0.115$ s, the wave crest touched to
 222 the top of the pipe and then filled the whole cross-section of the pipe at this point.



223
224 Fig. 3. Gas and liquid phase distribution in the terrain-inclined pipes

225 The distribution of velocity magnitude and static pressure near the elbow are
226 shown in Fig. 4. It represents that the gas velocity obviously increased when the gaseous
227 phase flows through the region above the liquid, and the maximum velocity can reach
228 36 m/s at $t = 0.005$ s. The gas flow area decreased due to the liquids assembled at the
229 bottom of the pipe, which caused the increase of the gas velocity when the gaseous
230 phase went through the region. At $t = 0.085$ s, the gaseous phase flowed through the
231 top of the wave crest and then it was accelerated. At $t = 0.115$ s, the distribution of
232 velocity in the pipe was different from the first two time steps due to the gaseous phase
233 was separated to two discontinuous regions by the liquid phase.

234 The pressure difference appeared around the elbow of the terrain-inclined pipe at
235 $t=0.005$ s due to the increase of the gas velocity. The gas-liquid phase interface was
236 disturbed and then the wave crest formed due to the large suction force overcoming the
237 effect of the gravity. At $t = 0.085$ s, the pressure difference region moved to the

238 downstream of the elbow and the suction force was directed to this region. The wave
 239 crest correspondingly continued to become larger and moved to the downstream of the
 240 elbow. At $t = 0.115$ s, some amount of the liquids filled the whole cross-section of the
 241 pipe and the gaseous phase was divided into two parts. The upstream pressure is higher
 242 than the downstream one. The liquids moved by the action of the pressure difference
 243 between the upstream and downstream.

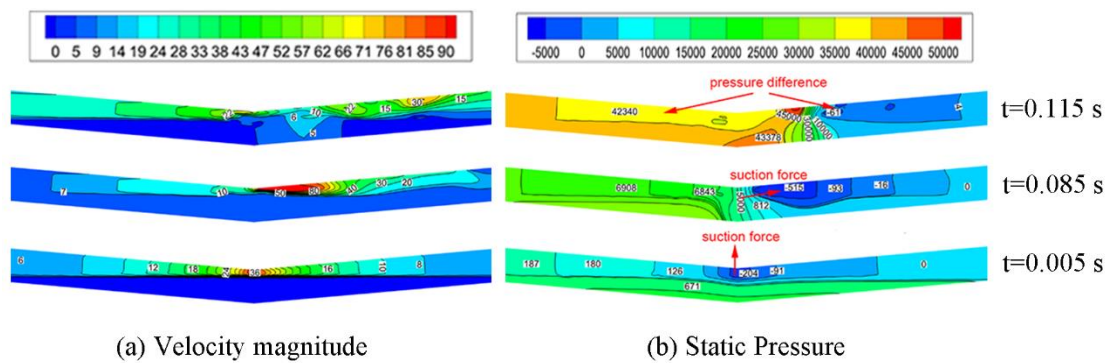


Fig. 4 Velocity and static pressure in the terrain-inclined pipes

4.3 Liquid motion along the uphill section

The gaseous and liquid phase distributions along the uphill section are shown in Fig. 5. The blue and red contours present the gaseous and liquid phase, respectively. We can see that the liquid phase blocked the pipe cross-section like a water plug after the liquid touched to the top of the pipe. It continued to move under the action of the pressure difference between the upstream and downstream, and some liquids moved along the top of the pipe and fell under the action of the gravity ($t = 0.12$ s - 0.40 s). At $t=0.80$ s - 5.0 s, the liquid motioned to the pipe outlet and some liquids flowed out the pipe exit under the action of the inertia and the shearing force of the gaseous phase. At $t=6.0$ s - 9.8 s, the remanent liquids refluxed to the elbow and the gravity played a dominant role in this fluid flow process.

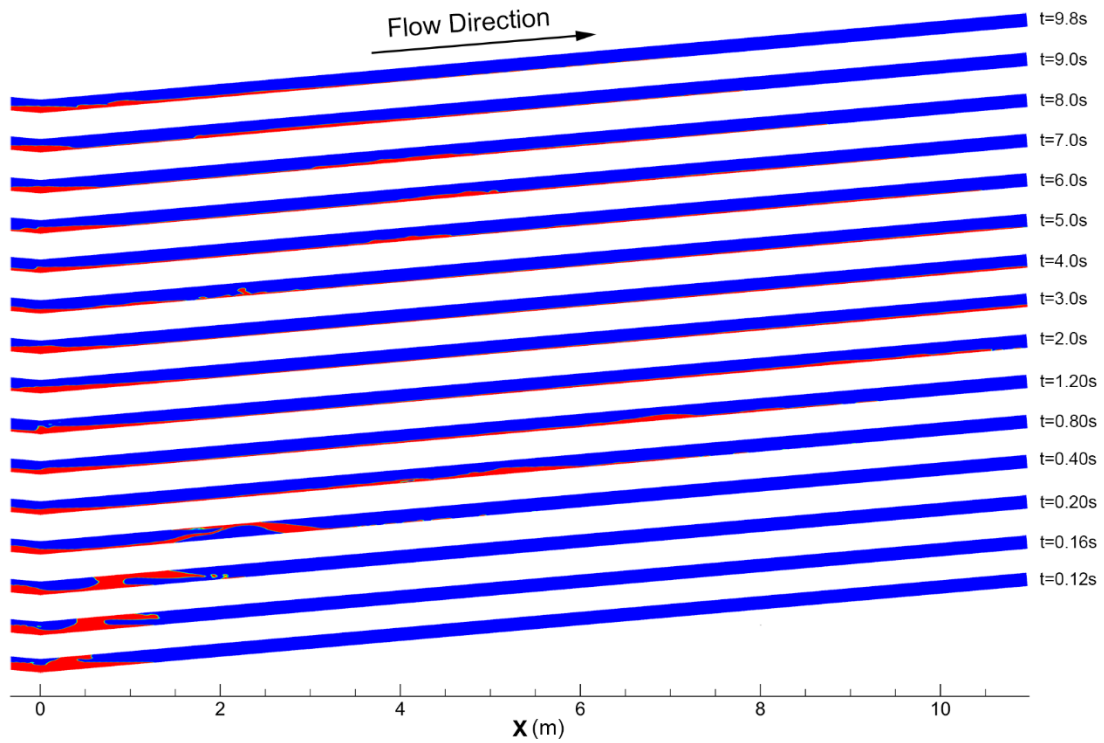


Fig. 5 Gas and liquid phase distribution along the uphill pipe

Fig. 6 shows the flow rate of the liquid phase at four cross-sections of the uphill pipe ($x = 1000$ mm, 3000 mm, 6000 mm, 7000 mm) under the superficial gas velocity of $U_G = 5.5$ m/s. The flow rate presented the periodic alternation between the positive and negative in Fig. 6 (a) - (d). The positive flow rate means that the liquid motioned along the uphill section, while the negative flow rate represents the liquid refluxed. In Fig. 6 (c), the flow rate is equal to 0 kg/s in this cross-section ($x = 6000$ mm) at $t = 11$ s - 17 s, and then a small amount of the liquid flowed through this cross-section of the uphill pipe. It illustrates that the liquid discontinuously flows through this pipe cross-section. We can see from the Fig. 7 (d) that there was almost no liquid going through this pipe cross-section ($x = 7000$ mm) after $t = 10$ s. We, therefore, can conclude that the periodic reflux occurs between the elbow and the pipe cross-sections less than $x = 6000$ mm.

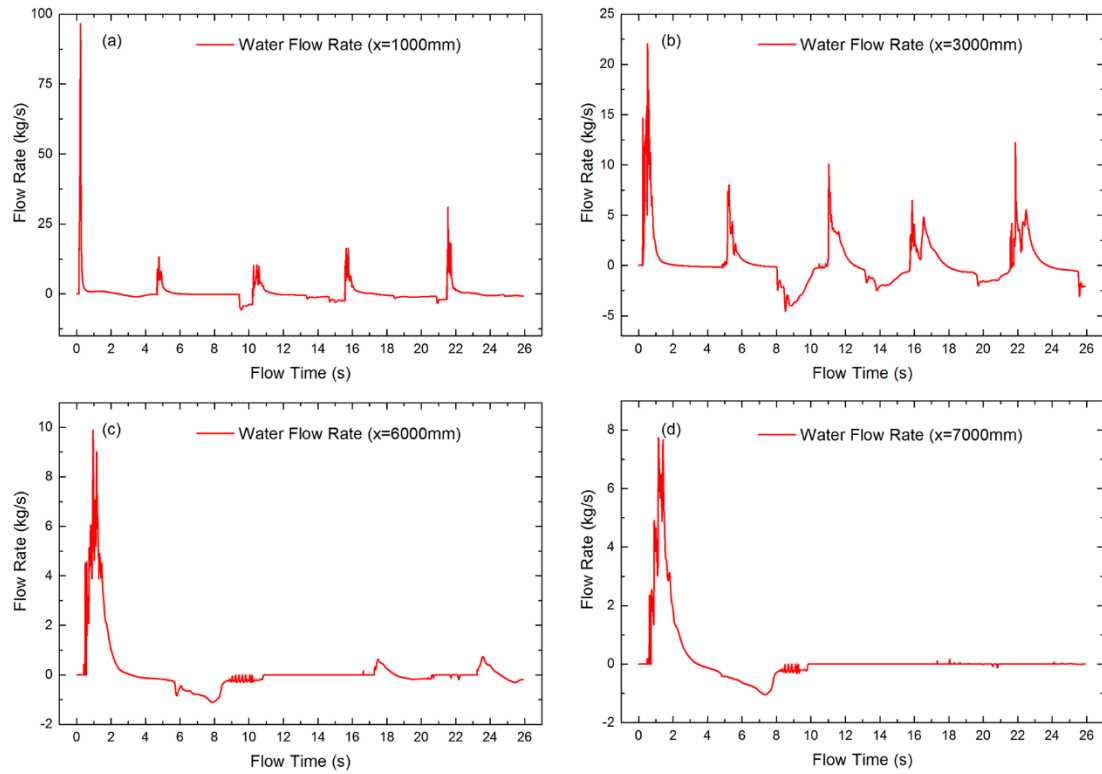


Fig. 6 Liquid flow rate in different cross-sections of the uphill pipe

Fig. 7 shows the liquid holdup in different cross-sections of the uphill pipe under the superficial gas velocity of $U_G = 5.5$ m/s. The fluctuation of the liquid holdup is shown in Fig. 7 (a) - (d). The liquid holdup showed the maximum peak of 1.0 in Fig. 7 (a). In other words, the liquids filled the whole cross-section of the uphill pipe at this point. As shown in Fig. 7 (b), the minimum peak of the liquid holdup was around 0.17 and the highest one was close to 0.40. In Fig. 7 (c) - (d), the maximum liquid holdup could reach 0.14 and the others were below 0.05. The average thickness of the liquid layer tended to become thinner from the uphill cross-section of $x = 1000$ mm to $x = 7000$ mm. The shearing force of the gaseous phase on the liquid was gradually weakened, and the liquids no longer flowed out the pipe exit.

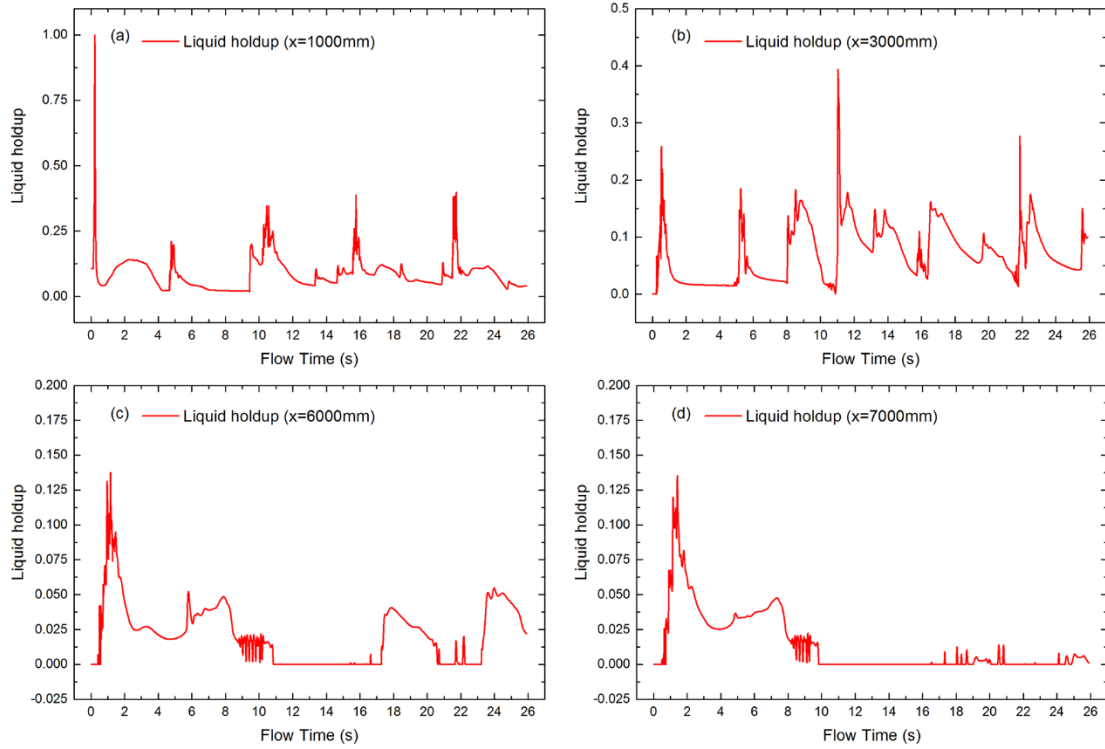


Fig. 7 Liquid holdup in different cross-section of the uphill pipe

The volume of the liquid was monitored in the numerical simulation and the results were presented in Fig. 8. The similar trends were obtained under three different inlet gas velocities. It can be seen that some liquids flowed out the pipe exit and the volume of liquid rapidly decreased in about 2 seconds. Later, the volume of the liquid was no longer changed. The volume percentages of the outflow liquid are approximately 20.16%, 28.4%, 31.58% under the superficial gas velocity of $U_G = 5.5$ m/s, 6.5 m/s, 7.5 m/s, respectively.

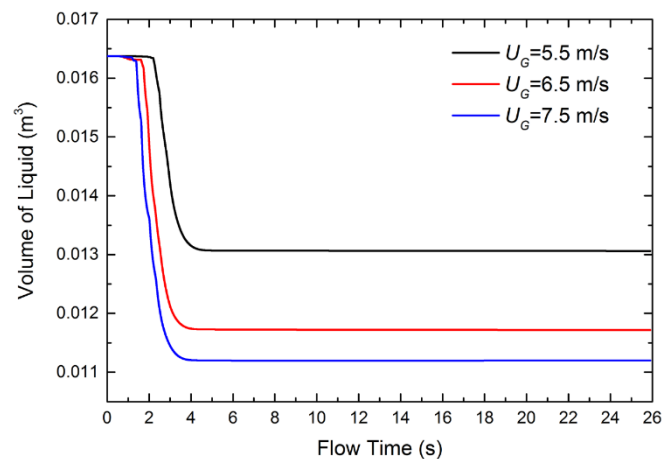


Fig. 8 Volume of the liquids in the terrain-inclined pipe

5. Conclusions

The VOF method and the RNG $k-\varepsilon$ turbulence model were used to simulate the gas-liquid two-phase flows within the terrain-inclined pipelines with deposited liquids. The flow area decreased because of the liquids gathered at the bottom of the pipe. It caused the increase of the gas velocity and the decrease of the static pressure, which generated the suction force above the gas-liquid interface. A wave crest formed and touched the top of the pipe under the action of the suction force around the elbow. The liquid blocked the cross-section of the pipe and correspondingly resulted in a large pressure drop. A certain amount of liquids can flow out the pipe exit carried by the gaseous phase. The volume percentages of the outflow liquids were 20.16%, 28.4%, 31.58% under the superficial gas velocity of 5.5 m/s, 6.5 m/s, 7.5 m/s, respectively. The liquid periodically flowed and returned along the uphill section in the pipe when the liquid was no longer flowing out the pipe exit.

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