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1 **A computational study of the EN 1078 impact test for bicycle helmets using a realistic**  
2 **subject-specific finite element head model**

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15

16 **A computational study of the EN 1078 impact test for bicycle helmets using a realistic**  
17 **subject-specific finite element head model**

18 **Abstract**

19 In this paper, the authors aim to establish a numerical framework for computational injury  
20 assessment of protective headgear subjected to drop tests. The case study is a bicycle helmet  
21 considered in a guided free fall impact test in accordance with the EN 1078 standard for  
22 certification of bicycle helmets. A finite element model of a bicycle helmet is created based on  
23 generic values for material properties and thicknesses of the helmet layup. The helmet model  
24 is coupled to an experimentally validated subject-specific finite element head model. This  
25 together constitute the numerical framework, which is found to be adequate by verification  
26 against other published results. The impact scenario is simulated with and without a bicycle  
27 helmet, and it is demonstrated that the helmet reduces peak resultant translational acceleration,  
28 von Mises skull stresses, intracranial pressure, and strain in the brain during the impact.

29 **Keywords:** head injury; bicycle helmet; drop test; biomechanics; crushable foam

30 **1 Introduction**

31 With increasing public awareness of health, fitness, and air pollution, cycling is popular as an  
32 alternative to the automobile. In countries like the Netherlands and Denmark, there are  
33 approximately 5 million and 1 million cyclists, or about 31% and 19% of the respective  
34 population, who prefer bicycles for commutation (European Commission, 2011).  
35 Nevertheless, cycling has associated safety risks. Cyclists are generally considered vulnerable  
36 road users, whereby a minor misjudgement by a vehicle driver can have significant adverse  
37 consequences on a cyclist. As opposed to other modes of commutation, cyclists only have their  
38 helmets for protection against fatal injuries in collisions. Therefore, it is particularly important

39 that bicycle helmets follow proven design standards.

40 Bicycle helmet standards generally require that the helmet manufacturers test and qualify their  
41 products by performing guided free fall drop tests using artificial headforms. Peak headform  
42 accelerations obtained in these tests are then compared against the thresholds prescribed in the  
43 standards. For instance, in the United States, the Consumer Product Safety Commission  
44 (CPSC, 1998) requires the registered headform peak accelerations to be below 300G for both  
45 a 2 m drop (pre-collision velocity of 6.3 m/s) on a flat anvil and 1.2 m drop (4.9 m/s) on  
46 hemispheric and curbstone anvils. In Europe, Asia, and South America, the acceleration  
47 threshold is 250G for both a 1.5 m drop (5.42 m/s) on a flat anvil and a 1.06 m drop (4.57 m/s)  
48 on a kerbstone anvil. Other standards include AS/NZS 2063 (Australia), CAN/CSAD1 13.2-M  
49 (Canada) and JIS T 8134 (Japan). Readers are referred to HEADS-ITN (2015) for a  
50 comprehensive overview of the current helmet standards.

51 There are currently 28 countries with some form of law for bicycle helmet use (ITF, 2017;  
52 European Commission, 2015, 2016; Bicycle Helmet Safety Institute, 2017). Moreover,  
53 approximately half of the members of the Organisation for Economic Co-operation and  
54 Development (OECD), the International Traffic Safety and Data Analysis Group, and the  
55 European Union have some form of bicycle helmet legislation. Wearing a helmet while cycling  
56 has been proven as an effective way to mitigate head injuries in relation to collisions (bicycle  
57 crashes or falls). Using headform impact tests, Cripton et al. (2014) showed that in a collision  
58 at 6.3 m/s and 5.42 m/s, severe brain injury is extremely likely to occur, with 99.9% probability  
59 based on the head injury criterion (HIC). However, these probabilities were reduced to 30.6%  
60 and 9.3% respectively, for the two impact velocities, when a bicycle helmet is worn. A similar  
61 study was published by McIntosh et al. (2013), who conducted impact tests with impact speeds  
62 of up to 6.9 m/s in several different impact orientations and locations. Here it was concluded

63 that helmet use was the most significant factor in reducing the probability of head injury. The  
64 effect has also been documented through epidemiological studies. Olivier and Creighton  
65 (2017) recently completed the largest ever systematic review and meta-analysis by collecting  
66 data from 40 different studies from USA, Europe, Australia and Asia. With data based on more  
67 than 64,000 injured cyclists, it was concluded that “helmet use is associated with odds  
68 reductions of 51% for head injury, 69% for serious head injury, 33% for face injury and 65%  
69 for fatal head injury”.

70 In addition to the aforementioned experimental and retrospective studies, computer  
71 simulations, such as the finite element (FE) method, provides an alternative to experimental  
72 methods in estimating the biomechanical responses of the human head. For example, the effect  
73 of utilising a bicycle helmet has been demonstrated with FE simulations of three different  
74 impact scenarios in Fahlsted (2016)’s study. This study showed that the risks of skull fracture  
75 and concussion were reduced by up to 98% and 46%, respectively, by wearing a helmet.  
76 Computer simulations for design and optimisation of protective headgear is also an active  
77 research field. Potentially, with the growing knowledge about head injuries and capabilities of  
78 modern computers, the level of protection can be accessed and analysed using FE simulations.  
79 For instance, Mills and Gilchrist (2008) employed a validated FE model of a commercially  
80 available bicycle helmet to investigate the effect of friction between the helmet shell and road  
81 surface on peak head acceleration in oblique impacts. The effect of foam liner material  
82 properties has been investigated by Asiminei et al. (2009), who varied both the density and  
83 modulus of the protective foam (expanded polystyrene, EPS). Teng et al. (2013) studied the  
84 effect of IMPAXX foam on head impact accelerations, as an alternative to the most commonly  
85 used foam type in bicycle helmets (EPS). As outlined, previous research has focused primarily  
86 on the helmet materials as well as helmet liners, but limited attention has been paid on head  
87 injury assessment. Moreover, many of these computational studies have used oversimplified

88 FE head models or artificial headforms for the head impact analyses which neither provide  
89 insights to the injury mechanics within the brain nor detail the brain regions which could be  
90 most affected by the impact.

91 The current study employs an anatomically detailed and experimentally verified subject-  
92 specific head model to simulate the particular impact prescribed by the helmet standard EN  
93 1078 for Europe, Asia, and South America. This computational approach does not only provide  
94 an alternative to experimental helmeted drop tests used in bicycle helmet test standards, but  
95 also allows injury assessment and evaluation of the intracranial biomechanical injury metrics,  
96 such as intracranial pressure (ICP) and strains, which cannot be determined using the headform  
97 or simplified FE head models, in these dynamic events. Finally, this study also investigates the  
98 effectiveness of a bicycle helmet in protecting our heads.

## 99 **2 Methods and materials**

### 100 ***2.1 Head model***

101 Geometrical information of the human skull and brain were obtained from high resolution axial  
102 computed tomography (CT) and magnetic resonance imaging (MRI) images of a 51-year-old  
103 Caucasian male subject respectively. These medical images were imported into Mimics v13.0-  
104 v14.0 (Materialise, Leuven, Belgium) for the segmentation and reconstruction of the FE human  
105 head model, which comprises the skeletal skull, nasal septal cartilage, nasal lateral cartilage,  
106 with the overlying soft tissue, the cerebrospinal fluid (CSF), the white and grey matters of  
107 cerebrum, cerebellum, the ventricular system, the midbrain, the brainstem as well as the air-  
108 containing sinuses. Various components of the head model can be seen in Figure 1. All the  
109 skeletal tissues such as cartilages and cervical vertebrae were modelled as linear elastic,  
110 isotropic materials while the brain tissues were assumed to be linear viscoelastic. The material  
111 properties of the various components of the FE head model can be found in Table 1 and Tse et

112 al. (2014). It should also be noted that the FE head model was validated against the ICP and  
113 relative displacement data of three cadaveric experiments (Nahum et al. 1977; Trosseille et al.,  
114 1992; Hardy, et al., 2001). More details on the development and validation of the FE head  
115 model can be found in Tse et al. (2014).

116 [Figure 1 around here]

117 [Figure 2 around here]

118 [Table 1 around here]

## 119 **2.2 Bicycle helmet model**

120 The FE model of the bicycle helmet was constructed based on geometry and thicknesses of a  
121 generic bicycle helmet layup. In addition, common helmet features such as ventilation holes  
122 and retention straps were incorporated in the helmet FE model (Figure 2). The full breakdown  
123 of properties and element types are listed in Table 1.

124 The bicycle helmet consists of five parts: (i) An energy absorbing EPS foam partly covered by  
125 (ii) a polycarbonate (PC) shell, with (iii) retention straps, (iv) cushion pads, and (v) a fixture to  
126 fit the helmet to the head model. All the components of the bicycle helmet, besides the EPS  
127 foam, were considered to be isotropic and linear elastic, and their material properties were  
128 taken from previously reported studies (Teng et al., 2013; van den Bosch, 2006; Tan et al.,  
129 2012) as shown in Table 1. Following the modelling approaches adopted by Mills and Gilchrist  
130 (2008) and Milne et al. (2012, 2014), the crushable foam plasticity model in Abaqus<sup>TM</sup>  
131 (Dassault Systemes, 2013) was used to mimic the EPS foam in this study. In addition, the  
132 crushable foam model employed in the study was validated experimentally by Cui et al. (2009)  
133 for EPS under impact loads. The constitutive law of this closed cell foam can be described in

134 three stages:

135 (i) In the first stage, the material response is linear elastic until the stress state reaches the  
136 yield stress.

137 (ii) In the second stage, the material reaches a plateau level where it continues to deform  
138 under an almost constant stress state. This corresponds to the level where most of the  
139 energy is absorbed as the foam cells collapse.

140 (iii) In the final stage, all foam cells have collapsed, and the material response is again  
141 linear elastic.

142 This constitutive law is shown in Figure 3 for the plastic range. According to Dassault Systemes  
143 (2013), the yield surface, which is shaped like an ellipse, is described by the following  
144 equations (1):

$$145 \sqrt{\sigma_Y + \alpha(p - p_0)^2} - B = 0, \text{ with } p_0 = \frac{p_c - p_t}{2}, \quad B = \alpha \frac{p_c + p_t}{2}, \quad \alpha = \frac{3k}{(3k_t + k)(3 - k)} \quad (1)$$

146 where  $\sigma_Y$  is the Von-Mises stress,  $p$  is the hydrostatic pressure,  $p_c$  and  $p_t$  are the yield strengths  
147 of the material in hydrostatic compression and tension, respectively.  $k$  describes the ratio  
148 between the initial yield stress in uniaxial compression and initial yield stress in hydrostatic  
149 compression, and  $k_t$  is the ratio between yield strength in hydrostatic tension and initial yield  
150 stress in hydrostatic compression. These values,  $k$  and  $k_t$ , were set to 0.1 and 1.933 (Table 1),  
151 respectively, according to Cui et al. (2009), Mills and Gilchrist (2008), and Dassault Systemes  
152 (2013).

153 [Figure 3 around here]



154 **2.3 Numerical replication of the EN 1078 standard (contact, boundary conditions, and**  
155 **loading)**

156 According to the EN 1078 standard, the “to be qualified” helmet equipped with the testing  
157 headform undergoes a guided free fall either using a twin wire or monorail test rig onto a flat  
158 anvil fixed on a rigid base. In our simulations, an initial velocity of 5.42 m/s corresponding to  
159 the final velocity before impact for an object subjected to gravity ( $G \equiv 9.81 \text{ m/s}^2$ ), was applied  
160 to the model, before impacting a fixed anvil (Figure 4). All the nodes at the bottom of the anvil  
161 were assigned fixed boundary conditions. The entire simulated impact was captured in 6 ms. It  
162 should be noted that the pretension in the retention straps of the bicycle helmet was accounted  
163 for in an initial step of the simulations prior to the impact, with a preloading of approximately  
164 5 N in each strap according to Mills and Gilchrist (2008).

165 The interaction between intracranial contents (skull and CSF, CSF and brain tissues as well as  
166 self-interaction of intracranial brain tissues) were represented by normal pressure-overclosure  
167 and tangential sliding contact definitions, with the coefficient of friction of 0.2 (Kleiven and  
168 Hardy, 2002; Willinger et al., 1995; Zhang et al., 2001), while the other extracranial bony and  
169 cartilaginous components are fused together. For the interfaces between the helmet and head,  
170 and helmet and anvil, contact was defined using a penalty formulation for tangential behaviour  
171 with kinetic friction. A kinetic friction coefficient of 0.2 was applied between the head and  
172 helmet, and a coefficient of 0.1 between the helmet and anvil in accordance with Milne et al.  
173 (2012). Similar to the modelling approaches adopted by Mills and Gilchrist (2008) and Milne  
174 et al. (2012, 2014), all the interfaces between different components of the helmet were meshed  
175 in such a way that they share common nodes with their adjacent interfaces. This technique is  
176 beneficial in terms of reducing computation time and is considered acceptable as no or very  
177 little relative motion between individual helmet components is expected. This implies that the

178 adhesion between components is considered to remain intact during the impact, meaning the  
179 collapsible EPS foam is the only helmet component that will absorb energy through irreversible  
180 deformation.

#### 181 ***2.4 Evaluation of head kinematic response and injury metrics***

182 The kinematic response of the head undergoing the impact was evaluated by its translational  
183 accelerations. These were calculated as mean values of the ten nodes nearest the centre of  
184 gravity of the head (marked as CG in Figure 1). For assessment of skull and brain injuries,  
185 three biomechanical metrics were evaluated to analyse the post-processed transient brain  
186 response. *Intracranial Strains*: Maximum principal strains were used as a measure of diffuse  
187 axonal injury (DAI). Morrison et al. (2003)'s in vitro tests of living brain tissue showed that a  
188 mechanical deformation with a positive principal strain exceeding 0.2 led to cell damage and  
189 death, and thus consequently DAI. *Intracranial Pressure (ICP)*: ICP has been hypothesised to  
190 be one of the most critical biomechanical injury metrics in traumatic brain injury (TBI) since  
191 an elevated ICP in head trauma can lead to severe brain damage (Tse et al., 2017). An ICP  
192 injury criterion, proposed by Ward et al. (1980) based on combined numerical and  
193 experimental investigation of live animals and human cadavers, states that serious or fatal brain  
194 injury occurs when the peak ICP exceeds 0.235 MPa, while no or minor brain injury occurs  
195 when the ICP is below 0.173 MPa. *Von Mises skull stress*: The critical failure level for the  
196 human skull is approximated by a von Mises stress threshold of 75 MPa, which is based on the  
197 average of the reported range from McElhaney et al. (1970)'s mechanical experiments  
198 conducted on human skull samples.

### 199 **3. Results and discussion**

200 Figure 4 shows the transient accelerations measured at the CG of the head, while Figure 5

201 illustrates the contour plots of the ICP and principal true strain for the brain, as well as the von  
202 Mises stresses of the skull at various time instants during the impact. The peak values are listed  
203 in Table 3.

204 [Figure 4 around here]

205 [Figure 5 around here]

206 [Table 3 around here]

### 207 ***3.1 Verification of the helmet model and numerical framework***

208 The peak translational acceleration and impact duration obtained in the current work were  
209 benchmarked against Milne et al. (2012)'s and Fahlstedt et al. (2016)'s FE modelling studies,  
210 which reported values for the same impact scenario in accordance with the EN 1078 standard.  
211 The comparison of the value of these parameters between the present work and the two  
212 aforementioned studies is presented in Table 2.

213 The predicted maximum head acceleration was found to be 223G, which was approximately  
214 28% and 1% higher than the values in Milne et al. (2012)'s and Fahlstedt et al. (2016)'s studies,  
215 respectively. As for the impulse duration, our simulation predicted that the impact generated  
216 an impulse lasting for 5 ms, while the respective impulse durations in Milne et al. (2012)'s and  
217 Fahlstedt et al. (2016)'s simulations were approximately 3 ms and 1 ms longer. The predictions  
218 presented in the current study are in good agreement with those values reported by Fahlstedt et  
219 al. (2016), even though the peak resultant translational acceleration appears to be overestimated  
220 and the impact duration to be underestimated when compared to Milne et al. (2012)'s  
221 simulations. These slight differences or discrepancy with Milne et al. (2012)'s study are  
222 justifiable due to the different head masses used in these simulations. As shown in Table 2, the

223 heavier head model used in Milne et al. (2012)'s simulation is expected to result in a more  
224 gradual velocity reversal, and thus a lower acceleration due to mass inertia effect. Moreover,  
225 our predicted peak head acceleration was also found to be within the 250 G threshold given by  
226 EN 1078 standard for direct impact, which implied that the helmet model adequately  
227 exemplifies a general bicycle helmet designed according to the EN 1078 standard.

228 [Table 2 around here]

### 229 ***3.2 Injury assessment using biomechanical metrics***

230 A peak in von Mises skull stress of 85 MPa was obtained at the impact site, shortly after the  
231 impact, as shown in Figure 5 and Table 3. This level is above the 75 MPa criterion based on  
232 McElhaney et al. (1970), which implies that the impact will result in a skull fracture. When  
233 wearing a helmet, the peak von Mises skull stress was reduced to approximately 10 MPa, which  
234 corresponds to a reduction of 88%. As this level was well below the failure limit of 75 MPa,  
235 this demonstrates that the bicycle helmet is effective in preventing skull fracture for this  
236 particular impact condition.

237 The maximum principal true strain of the brain occurred after the peak in von Mises stress of  
238 the skull, due to a natural delay for the motion of the softer viscoelastic brain. The region near  
239 the impact zone at the parietal skull bone experienced relative high strains of about 0.8, which  
240 was reduced to 0.7 (-13%) by equipping with a helmet. As shown in Figure 5, there were large  
241 areas of the brain exceeding Morrison et al. (2003)'s strain injury threshold of 0.2 during the  
242 simulated impact duration. This indicates that there is a significant risk of DAI in both impact  
243 scenarios, which is however reduced when a helmet is worn as the strain levels are lower.

244 The intracranial pressure (ICP) response behaved similarly as the brain principal strains. The  
245 maximum positive ICP, at 0.65 MPa, was observed in regions close to the impact zone mainly

246 in the grey matter, which was reduced to 0.45 MPa (31%) when a helmet was helmet worn.  
247 Both impacts resulted in values that exceeded Ward et al. (1980) ICP threshold of 0.235 MPa,  
248 but again a significant reduction was observed when the head model was equipped with the  
249 helmet.

### 250 **3.3 Limitations**

251 A key limitation of this study was that the intracranial transient responses, which are the basis  
252 for injury assessment using the proposed injury metrics, are not only dependent on the physical  
253 parameters employed, but also the interaction properties and constraints enforced in the model.  
254 As it appears in Figure 5, relative deformation occurred between the intracranial components  
255 of the head. In particular, intermittent gaps develop in interfaces between the skull and CSF,  
256 and the CSF and white matter during the impact. These are the potential locations for negative  
257 ICP which could also be an important biomechanical injury metric, but could also be caused  
258 by limitations of the model.

259 The pulsatile cerebrospinal fluid (CSF) was assumed linear elastic for the head model, which  
260 is a normal, necessary assumption for head models with this level of complexity. One cannot  
261 resort to computationally intensive fluid-structure interaction simulations, and this is usually  
262 acceptable due to limited relative motion between the brain components. Furthermore, the  
263 components in the current head model were unconstrained, except for the contact property  
264 between them which restrained penetration. In reality, the various head components are  
265 connected by different nerves, which restrict their relative motion to some extent. The  
266 discretionary use of different types of contact interactions or constraints would affect the  
267 simulation results, as also noted and reported by Bar-Kochba et al. (2012), but addressing this  
268 issue is complex since there is limited in-vivo human head test data to reveal the actual contact  
269 interaction properties between the various components.

270 Smaller regions of high ICPs and strains were observed in the brain during the impact as seen  
271 in Figure 5. When using a penalty contact formulation, penetration is avoided by upholding a  
272 restoring, coupling force between components. As for this relatively complex geometry of the  
273 brain and these large deformations, many local areas with high pressure can arise from these  
274 contact forces. This phenomenon was not reported by Milne et al. (2012) who employed a  
275 relatively simplified FE head model, in which the convoluted topography of the brain was  
276 idealised as a smooth surface.

#### 277 **4. Conclusion**

278 In the current study, an enhanced, anatomically detailed and experimentally verified subject-  
279 specific head model, coupled with a bicycle helmet model was used to simulate the test impact  
280 as prescribed by the helmet standard EN 1078 for Europe, Asia, and South America. Using the  
281 simulation model, it was demonstrated how the peak resultant translational acceleration was  
282 reduced by 75%, the von Mises skull stress by 88%, the ICP by 31%, and the max. principal  
283 strain in the brain by 15% when the model was wearing a bicycle helmet in the test impact. It  
284 is anticipated that the presented numerical model can be used in the future to go beyond the  
285 enforced test standards in terms of evaluating and optimising helmet designs by analysing  
286 multiple and more complex injury scenarios.

287

288 ***3609 words***

#### 289 **Conflicts of interest**

290 None

291 **Acknowledgments**

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295 **References**

- 296 Asiminei, A. G., Goffin, J., Van Der Perre, G., & Verpoest, I. (2009). A transient finite element  
297 study reveals the importance of the bicycle helmet material properties on head protection  
298 during an impact. *International Research Council on the Biomechanics of Injury - 2009*  
299 *International IRCOBI Conference on the Biomechanics of Injury, Proceedings*, 357–360.
- 300 Bar-Kochba, E., Gutttag, M., Sett, S., Franck, J. A., McNamara, K., Crisco, J., & Franck, C.  
301 (2012). Finite Element Analysis of Head Impact in Contact Sports. In *SIMULIA*  
302 *Community Conference (SCC)* (p. 10).
- 303 Bicycle Helmet Safety Institute (2017). Bicycle Helmet Laws. Available at  
304 <https://helmets.org/mandator.htm>.
- 305 Cui, L., Kiernan, S., & Gilchrist, M. D. (2009). Designing the energy absorption capacity of  
306 functionally graded foam materials. *Materials Science and Engineering: A*, 507(1), 215-  
307 225.
- 308 CPSC (1998), Consumer Product Safety Commission, Safety Standard for Bicycle Helmets;  
309 Final Rule, USA
- 310 Cripton, P. A., Dressler, D. M., Stuart, C. A., Dennison, C. R., & Richards, D. (2014). Bicycle  
311 helmets are highly effective at preventing head injury during head impact: Head-form  
312 accelerations and injury criteria for helmeted and unhelmeted impacts. *Accident Analysis*  
313 *& Prevention*, 70, 1-7.
- 314 Dassault Systmes (2013), ABAQUS User's Manual 6.13, Dassault Systmes, USA

315 European Commission (2011), The Gallup Organisation, Future of transport - Analytical  
316 report, Europe

317 European Commission (2015). Road safety in the European Union; Trends, statistics and main  
318 challenges, European Union

319 European Commission (2016) Road safety in the European Union; Trends, statistics and main  
320 challenges, European Union

321 European standard (1997), EN 1078, Helmets for pedal cyclists and for users of skateboards  
322 and roller skates

323 Kleiven, S., & Hardy, W. N. (2002). Correlation of an FE Model of the Human Head with  
324 Local Brain Motion--Consequences for Injury Prediction. *Stapp Car Crash Journal*, (46),  
325 123-44.

326 Fahlstedt, M., Halldin, P., & Kleiven, S. (2016). The protective effect of a helmet in three  
327 bicycle accidents—A finite element study. *Accident Analysis & Prevention*, 91, 135-143.

328 Hardy WN, Foster C, Mason M, Yang K, King A, Tashman S. Investigation of head injury  
329 mechanisms using neutral density technology and high-speed biplanar x-ray. *45th Stapp*  
330 *Car Crash Conference*, San Antonio, USA, 2001; 337–368.

331 HEADS-ITN (2015). *Current Standards for Sports and Automotive Helmets: A Review.*,  
332 Available at [http://www.heads-itn.eu/pdfs/Helmets\\_Standard\\_Evaluation.pdf](http://www.heads-itn.eu/pdfs/Helmets_Standard_Evaluation.pdf) (Section  
333 2.1)

334 ITF (2017). Road Safety Annual Report 2017, OECD Publishing, Paris.

335 McElhaney, J. H., Fogle, J. L., Melvin, J. W., Haynes, R. R., Roberts, V. L., & Alem, N. M.  
336 (1970). Mechanical properties of cranial bone. *Journal of biomechanics*, 3(5), 495-511.

337 McIntosh, A. S., Lai, A., & Schilter, E. (2013). Bicycle helmets: head impact dynamics in  
338 helmeted and unhelmeted oblique impact tests. *Traffic injury prevention*, 14(5), 501-508.



339 Morrison III, B., Cater, H. L., Wang, C. C., & Thomas, F. C. (2003). A tissue level tolerance  
340 criterion for living brain developed with an in vitro model of traumatic mechanical loading.  
341 *Stapp Car Crash Journal*, 47, 93.

342 Mills, N. J., & Gilchrist, A. (2008). Finite-element analysis of bicycle helmet oblique impacts.  
343 *International Journal of Impact Engineering*, 35(9), 1087-1101.

344 Milne, G., Deck, C., Carreira, R. P., Allinne, Q., & Willinger, R. (2012). Development and  
345 validation of a bicycle helmet: Assessment of head injury risk under standard impact  
346 conditions. *Computer methods in biomechanics and biomedical engineering*, 15(1), 309-  
347 310.

348 Milne, G., Deck, C., Bourdet, N., Carreira, R. P., Allinne, Q., Gallego, A., & Willinger, R.  
349 (2014). Bicycle helmet modelling and validation under linear and tangential impacts.  
350 *International Journal of Crashworthiness*, 19(4), 323-333.

351 Nahum AM, Smith R, Ward CC. Intracranial pressure dynamics during head impact. In *21st*  
352 *Stapp Car Crash Conference. Society of Automotive Engineers (SAE)*, SAE Paper No.  
353 770922: San Diego, USA, 1977; 339–366.

354 Newman, J. A., Shewchenko, N., & Welbourne, E. (2000). A proposed new biomechanical  
355 head injury assessment function-the maximum power index. *Stapp car crash journal*, 44,  
356 215-247.

357 Olivier, J., & Creighton, P. (2016). Bicycle injuries and helmet use: a systematic review and  
358 meta-analysis. *International journal of epidemiology*, 46(1), 278-292.

359 Schaller, A., Voigt, C., Huempfer-Hierl, H., Hemprich, A., & Hierl, T. (2012). Transient finite  
360 element analysis of a traumatic fracture of the zygomatic bone caused by a head collision.  
361 *International journal of oral and maxillofacial surgery*, 41(1), 66-73.

362 Takhounts, E. G., Eppinger, R. H., Campbell, J. Q., & Tannous, R. E. (2003). On the  
363 development of the SIMon finite element head model. *Stapp car crash journal*, 47, 107.

364 Tan, L. B., Tse, K. M., Lee, H. P., Tan, V. B. C., & Lim, S. P. (2012). Performance of an  
365 advanced combat helmet with different interior cushioning systems in ballistic impact:  
366 Experiments and finite element simulations. *International Journal of Impact Engineering*,  
367 50, 99-112.

368 Teng, T. L., Liang, C. L., & Nguyen, V. H. (2013). Development and validation of finite  
369 element model of helmet impact test. Proceedings of the Institution of Mechanical  
370 Engineers, *Part L: Journal of Materials Design and Applications*, 227(1), 82-88.

371 Trosseille X, Tarriere C, Lavaste F. Development of a FEM of the human head according to a  
372 specific test protocol. In *30th Stapp Car Crash Conference. Society of Automotive*  
373 *Engineers (SAE)*, SAE Paper No. 922527: Warrendale, USA, 1992; 235–253.

374 Tse, K. M., Tan, L. B. and Lee, H. P. (2017). Chapter 10: The Skull and Brain: Computer  
375 models for the head and its protection. In Franklyn, M. and Lee, P. V. S. (eds) *Military*  
376 *Injury Biomechanics: The Cause and Prevention of Impact Injuries*: CRC Press, pp. 175-  
377 220.

378 Tse, K. M., Tan, L. B., Lim, S. P., & Lee, H. P. (2015). Conventional and complex modal  
379 analyses of a finite element model of human head and neck. *Computer methods in*  
380 *biomechanics and biomedical engineering*, 18(9), 961-973.

381 Tse, K. M., Tan, L. B., Lee, S. J., Lim, S. P., & Lee, H. P. (2014). Development and validation  
382 of two subject-specific finite element models of human head against three cadaveric  
383 experiments. *International journal for numerical methods in biomedical engineering*,  
384 30(3), 397-415.

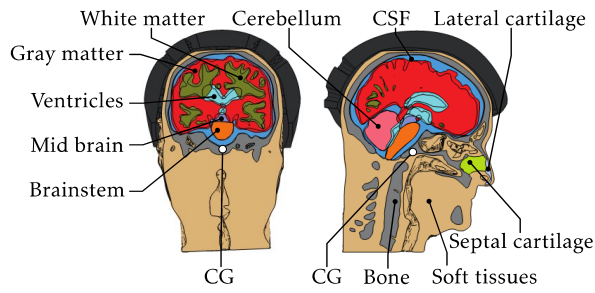
385 E. van den Bosch (2006), "Crash Helmet Testing and Design Specifications", Ph.D. thesis,  
386 Eindhoven University of Technology, Department of Mechanical Engineering, Section of  
387 Dynamics and Biomechanics.

388 Ward, C., Chan, M., & Nahum, A. (1980). Intracranial pressure—a brain injury criterion (No.  
389 801304). SAE Technical Paper.

390 Willinger, R., Taleb, L., & Kopp, C. M. (1995). Modal and temporal analysis of head  
391 mathematical models. *Journal of neurotrauma*, 12(4), 743-754.

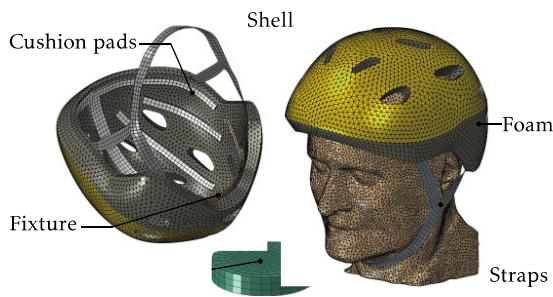
392 Zhang, L., Yang, K. H., & King, A. I. (2004). Comparison of brain responses between frontal  
393 and lateral impacts by finite element modeling. *Journal of neurotrauma*, 18(1), 21-30.

394



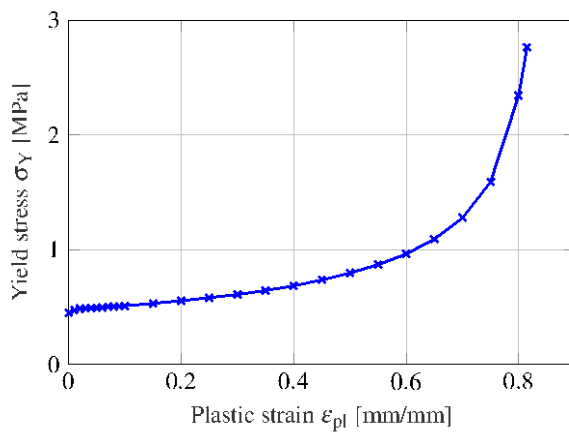
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396 Figure 1: Frontal and sagittal view of head model, and probe location (CG).



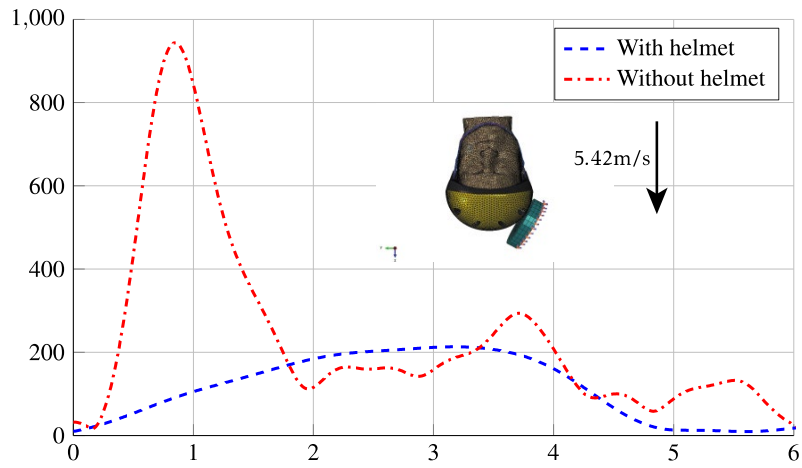
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398 Figure 2: Mesh of bicycle helmet model, anvil and the assembled helmet-head model with  
 399 orientation of the coordinate system.



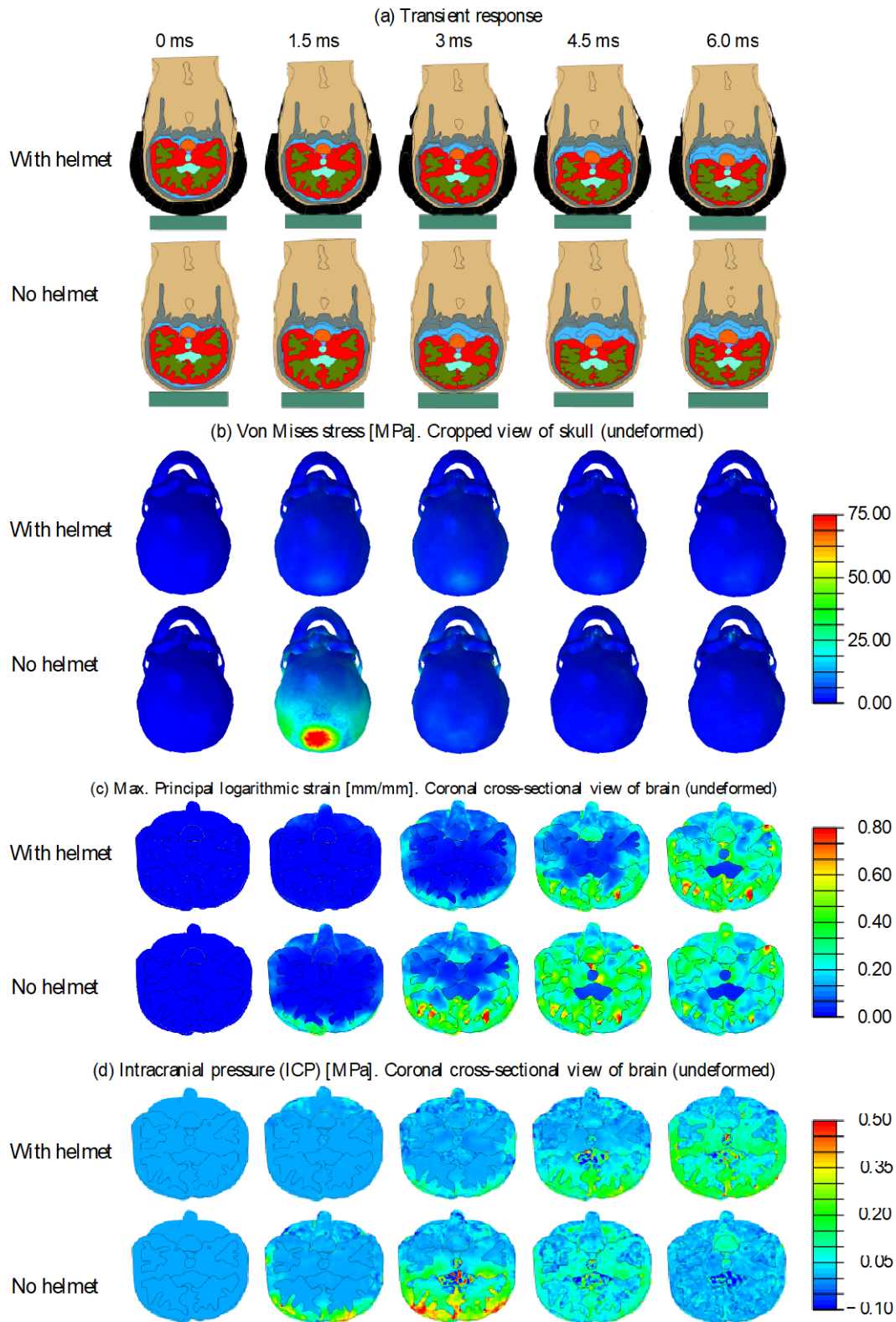
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401 Figure 3: Constitutive relation (Yield stress/plastic strain) of the expanded polystyrene (EPS)  
 402 foam.



403

404 Figure 4: Resulting translational acceleration at CG of the headform for the EN 1078 impact,  
 405 equivalent to a 1.5 m guided free fall.



406

407 Figure 5: Resulting field variables for the EN 1078 impact: Guided 1.5 m direct free fall on  
 408 flat anvil. (a) Transient response, (b) Von mises stress, (c) Principal logarithmic strain, and  
 409 (d) Intracranial pressure.

410

Components of the helmet	No. and types of elements *	Thickness [mm]	Elasticity [MPa]	Poisson's ratio [-]	Density [kg×mm <sup>-3</sup> ]	References
EPS Foam <sup>#</sup>	48,416 C3D4	20	E = 20; $\sigma_Y = 0.140$ ; k = 0.1; $k_t = 1.933$	-	$0.064 \times 10^{-6}$	Dassault Systmes (2013); Cui et al. (2009); Mills and Gilchrist (2008)
Cushion pads	1,632 C3D4	3	E = 0.47	-	$0.032 \times 10^{-6}$	Teng et al. (2013)
Shell	4,116 S3R	0.4	E = 15	0.42	$1.5 \times 10^{-6}$	van den Bosch (2006)
Fixture	128 S4R	1.5	E = 2	0.35	$1.5 \times 10^{-6}$	
Straps	342 S4R	1.5	E = 0.06	0.25	$0.4 \times 10^{-6}$	Tan et al. (2012)
<i>Total, helmet: 54,000 elements, 0.194 kg</i>						
Components of the test setup	No. and types of elements *	Thickness [mm]	Elasticity [MPa]	Poisson's ratio [-]	Density [kg×mm <sup>-3</sup> ]	References
Anvil	576 C3D8R	24	E = 200	0.25	$7.8 \times 10^{-6}$	European standard (1997)
<i>Total, anvil: 576 elements, 2.32 kg</i>						
Components of the head	No. and types of elements *	Elasticity / Viscoelasticity [MPa] $G(t) = G_\infty + (G_0 - G_\infty)e^{-\beta t}$ with $G_{0,\infty}$ [MPa], $\beta$ [s <sup>-1</sup> ])		Poisson's ratio [-]	Density [kg×mm <sup>-3</sup> ]	References
Brainstem <sup>^</sup>	6,104 C3D4	$G_0 = 0.0225$ , $G_\infty = 0.0045$ , $\beta = 80$		0.4996	$1.06 \times 10^{-6}$	Tse et al. (2014)
Cerebral peduncle <sup>^</sup>	1,762 C3D4	$G_0 = 0.0225$ , $G_\infty = 0.0045$ , $\beta = 80$		0.4996	$1.06 \times 10^{-6}$	Tse et al. (2014)
CSF	164,864 C3D4	E = 1.314		0.4999	$1.04 \times 10^{-6}$	Tse et al. (2014)
Grey matter <sup>^</sup>	436,917 C3D4	$G_0 = 0.034$ , $G_\infty = 0.0064$ , $\beta = 700$		0.4996	$1.04 \times 10^{-6}$	Tse et al. (2014)
Lateral cartilage	2,874 C3D4	E = 30		0.45	$1.50 \times 10^{-6}$	Tse et al. (2014)
Septum cartilage	3,578 C3D4	E = 9		0.32	$1.50 \times 10^{-6}$	Tse et al. (2014)
Bone	130,482 C3D4	E = 8000		0.22	$1.21 \times 10^{-6}$	Tse et al. (2014)
Soft tissues	253,894 C3D4	E = 16.7		0.46	$1.04 \times 10^{-6}$	Tse et al. (2014)
Ventricles	36,776 C3D4	E = 1.314		0.4999	$1.04 \times 10^{-6}$	Tse et al. (2014)
White matter <sup>^</sup>	278,925 C3D4	$G_0 = 0.034$ , $G_\infty = 0.0064$ , $\beta = 700$		0.4996	$1.04 \times 10^{-6}$	Tse et al. (2014)
<i>Total, head: 1,300,000 elements, 4.73 kg</i>						

<sup>#</sup> with material non-linearity, volumetric hardening

<sup>^</sup> with material non-linearity, viscoelasticity, where  $G_\infty$  is the long-term shear modulus,  $G_0$  is the short term shear modulus, and  $\beta$  is the decay factor

\* C3D4 refers to solid tetrahedral and C3D8R refers to solid hexahedral elements. S3R and S4R are reduced-integration triangular and quadrilateral shell elements, respectively.

411

412 Table 1: Mechanical properties for the different components. If not otherwise stated,

413 components are considered isotropic and linear-elastic.

<b>Reference</b>	<b>Head model</b>	<b>Peak resultant linear acceleration</b>	<b>Impact duration</b>
The current study	Tse et al. (2014), 4.73 kg	223G	5ms
Milne et al. (2012)	Strasbourg University Finite Element Head Model, 5.7 kg	174G	8ms
Fahlstedt et al. (2016)	Hybrid III dummy head, 4.54 kg	220G	6ms

414

415 Table 2: Benchmarks for model validation.

<b>Field variable</b>	<b>Without helmet</b>	<b>With helmet</b>	<b>Reduction</b>
Peak resultant linear acceleration	944G	223G	75%
Von Mises stress in skull	85 MPa (*1.2ms)	10 MPa (3.6ms)	88%
Max. Intracranial pressure (ICP) in brain	0.65 MPa (3.4ms)	0.45 MPa (5.3ms)	31%
Max. Principal strain in brain	0.8 [-] (3.4ms)	0.7 [-] (5.4ms)	15%

(\* all parentheses hold the time of the occurrence)

416

417 Table 3: Peak values of acceleration and field variables, incl. reductions.