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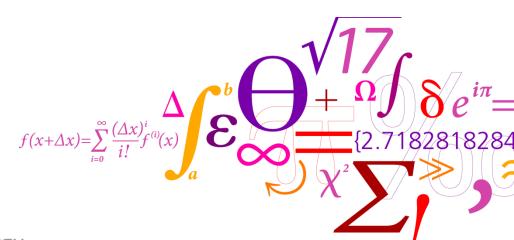
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Comparison of wind turbine wake properties in nonuniform inflow predicted by different CFD rotor models

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Risø DTU

National Laboratory for Sustainable Energy



Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)



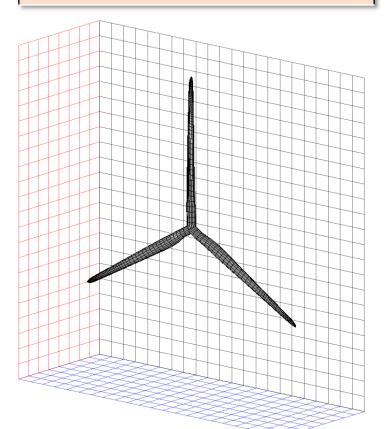




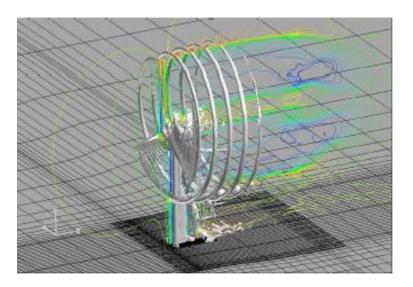


Wind turbine models in CFD

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- Actuator line model (AL)
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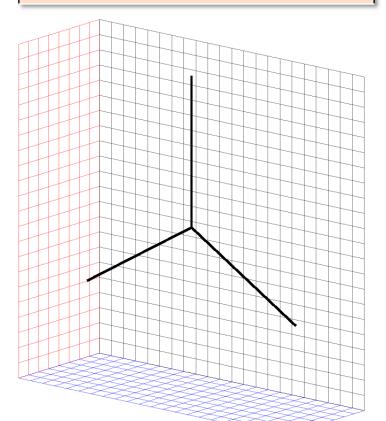
- The blade/airfoil boundary layer is resolved.
- \triangleright The required number of grid points for one rotor using RANS is O(10⁷)
- Provides detailed insight about flow behaviour
- Usually used for accurately predict loads and power production
- ➤ Too computationally heavy for several wind turbines.





Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)



Blades represented as lines.



Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

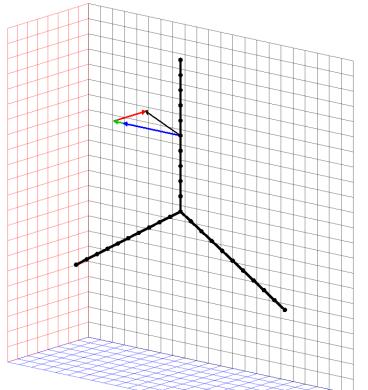
- Blades represented as lines.
- Aerodynamic blade forces determined from 2D airfoil data.

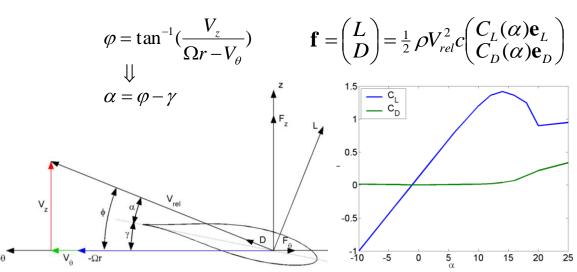


Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- Blades represented as lines.
- ➤ Aerodynamic blade forces determined from 2D airfoil data.





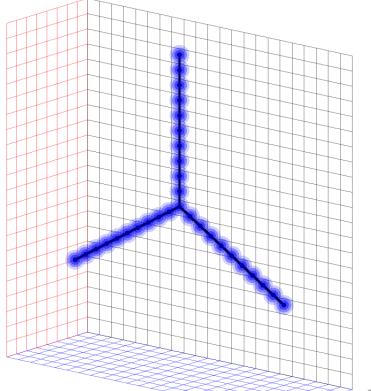


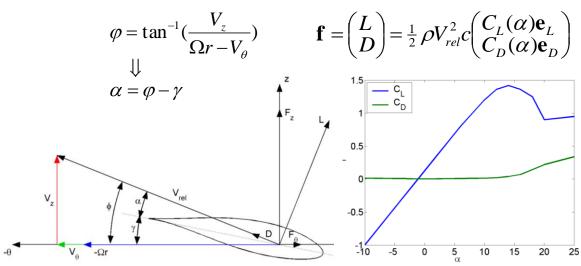
Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- Blades represented as lines.
- ➤ Aerodynamic blade forces determined from 2D airfoil data.
- ➤ Blade forces smeared to avoid singular behaviour. $\begin{bmatrix} & & & & & & \\ & & & & & \\ & & & & & \end{bmatrix}$

 $\mathbf{f}_{\varepsilon} = \mathbf{f} \otimes \boldsymbol{\eta}_{\varepsilon}, \quad \boldsymbol{\eta}_{\varepsilon} = \frac{1}{\varepsilon^{3} \pi^{3/2}} \exp \left[-\frac{d^{2}}{\varepsilon^{2}} \right]$







Wind turbine models in CFD

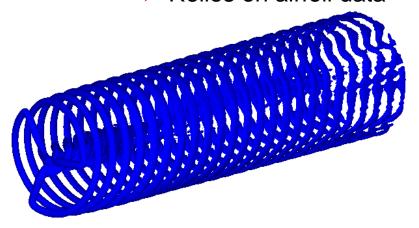
- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

Advantages:

- ➤ Low number of grid points O(10⁶) needed compared to full rotor CFD.
- Applicable with simple grid geometries.
- ➤ Captures the most important features of the wake including tip/root vortices.
- ➤ Well suited for LES simulations (no boundary layers need to be resolved)

Disadvantages:

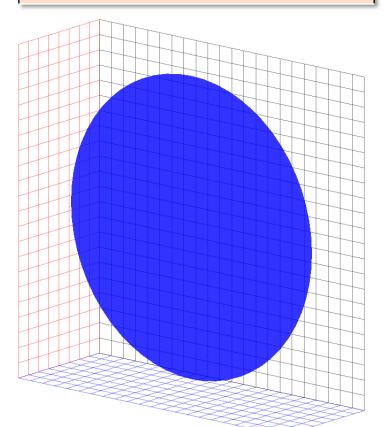
Relies on airfoil data





Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

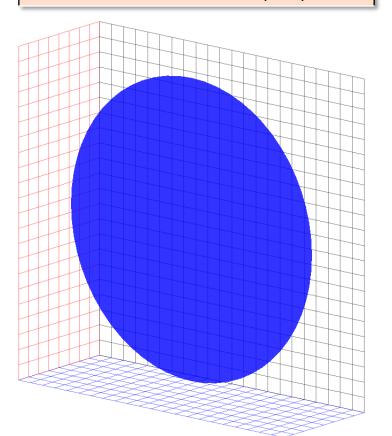


Rotor represented by forces distributed on permeable disc.



Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)



- Rotor represented by forces distributed on permeable disc.
- ➤ The disc loading is either prescribed or determined from airfoil data.
- Pressure velocity decoupling avoided using Gaussian force smearing or a modified Rhie-Chow algorithm



Wind turbine models in CFD

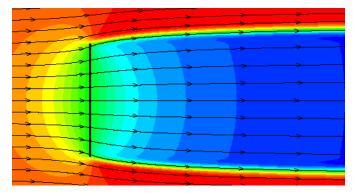
- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

Advantages:

- Low number of grid points
- Applicable with simple grid geometries
- Well suited for LES simulations
- Large time steps can be used
- Can run in steady state

Disadvantages:

- Relies on airfoil data
- Does not capture influence of individual blades
- May be questionable in non-uniform inflow



Axial velocity contours and streamlines for a uniformly loaded disc at C_T =0.89



Summary:

- AL/AD typically used for wake studies
- Details of rotor geometry assumed unimportant in far wake



Objectives:

- ➤ Study importance of wind turbine model on wake characteristics
 - ➤ How much details are lost due to the simpler models?



- Conduct a consistent comparison of the three models
 - Same numerical setup for all models

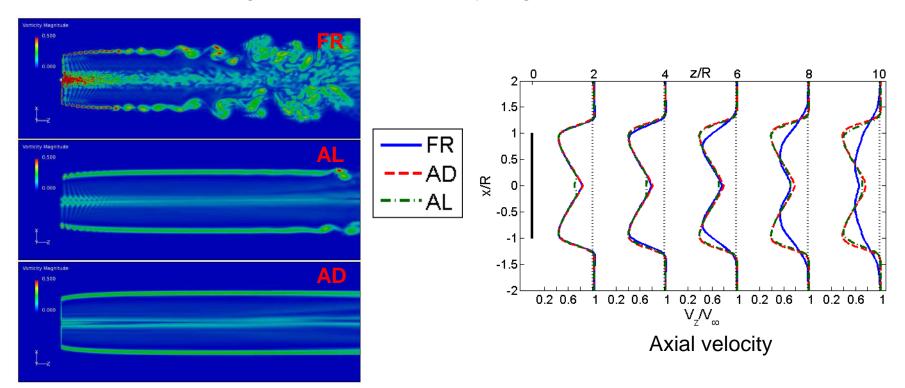


Previous work



Simulations of NREL 5MW reference turbine in non-sheared laminar inflow

- Wake of FR develops faster into a bell shaped form than the AL and AD.
- Faster spreading of wake is caused by larger TKE in the FR simulation.

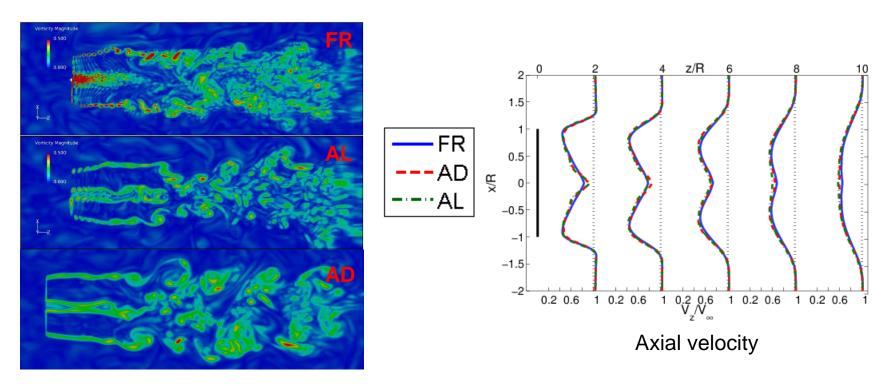


Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Previous work



Simulations of NREL 5MW reference turbine in non-sheared turbulent inflow



Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Present work



Objectives:

- ➤ Study importance of wind turbine model on wake characteristics in non-uniform inflow:
 - Sheared inflow
 - Yawed inflow
- ➤ Simulating the 2MW NM80 turbine using similar numerical setup







Approach – Flow solver



EllipSys3D:

- In-house CFD code
- Incompressible Navier-Stokes equations
- > Finite volume discretization
- Structured curvelinear grids
- Pressure/Velocity formulation
- Multigrid
- Multiblock
- Grid sequencing
- > MPI

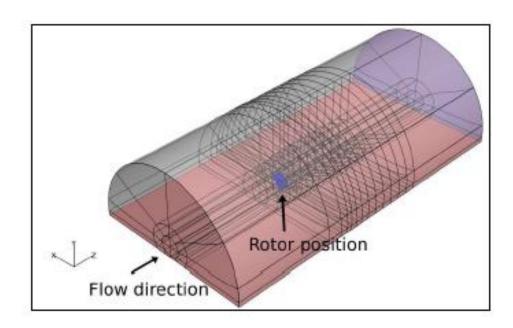
Solver parameters:

- > QUICK/QUICK_CDS4
- > SIMPLE
- > DES



Background mesh:

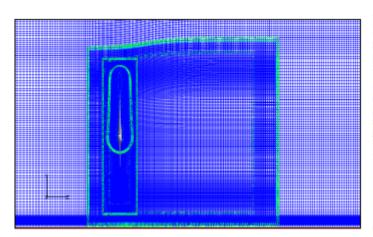
- Same background mesh for all simulations
- Half cylinder with radius 8D
- ➤ 308 blocks of 32³ (10.1 ·10⁶ cells)
- ➤ High resolution of the first 5D of the wake (cell size 1.3m x 1.3m x 0.8m)

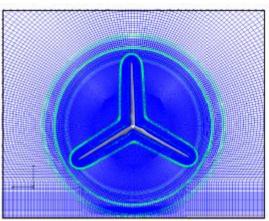




Full rotor with overset grid:

- Four overlapping mesh groups
- Rotor mesh generated using HypGrid3D to form an O-O topology
- ➤ Total number of grid points is 26.7·10⁶
- Rotor surface with a non-slip boundary condition
- \rightarrow First cell height y=1.0·10⁶ (y+ < 2)







Actuator line simulations:

- ➤ Same background mesh as the full rotor simulation (10.1 ·10⁶ cells)
- Force smearing using 3D convolution
- > 33 force elements along each line

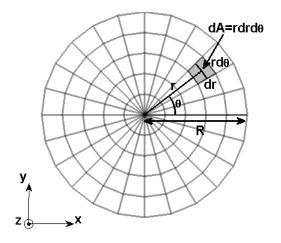


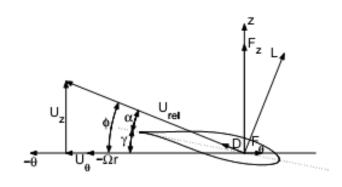


Actuator disc simulations:

- ➤ Same background mesh as the full rotor simulation (10.1 ·10⁶ cells)
- 33 radial force elements
- ➤ Force smearing using 1D convolution in normal and radial direction
- Forces on each differential area dA=rdrdθ is determined from local flow conditions and airfoil data.









- Sheared inflow
- Yawed inflow



- Sheared inflow
- Yawed inflow

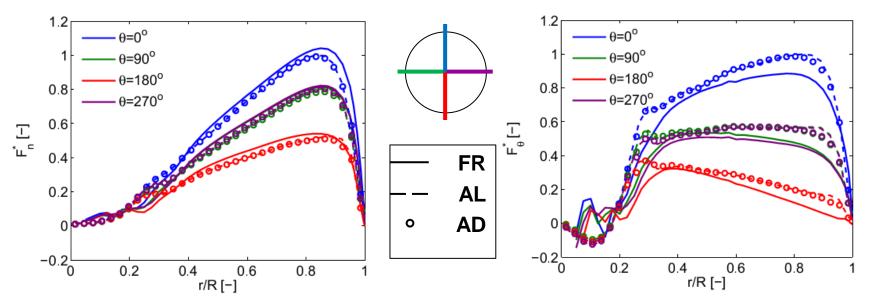
- $V_{\infty} = 8 \text{ m/s}$
- \triangleright Power law inflow ($\alpha = 0.55$):

$$V_Z = V_\infty \left(\frac{y}{H}\right)^\alpha$$



- Sheared inflow
- Yawed inflow

- Normal loads in good agreement
- Tangential loads less in FR than in AL and AD



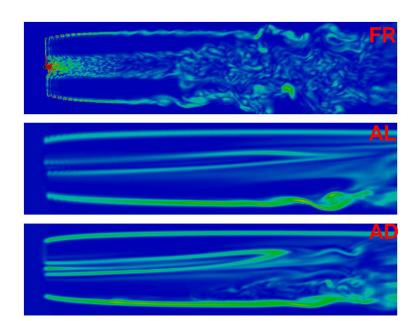
Spanwise distribution of normal and tangential loads at various azimuth positions



Test cases

- Sheared inflow
- Yawed inflow

- ➤ Vorticity from tip vortices much stronger in FR than in AL and AD.
- Wake of FR more unstable
- Similar vorticity contours for AL and AD (except for instability in the far wake)
- Reasons for more unstable wake of FR:
 - Higher grid resolution
 - > Fluctuating loads (e.g. stall effets near root)

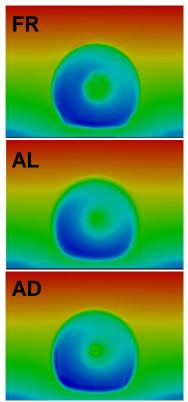


Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

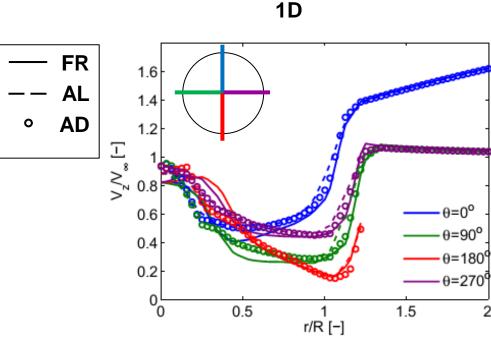


- Sheared inflow
- Yawed inflow

- Good agreement in predicted near wake deficit
- AL and AD in close agreement
- ➤ Wake of FR develops faster into a bell shaped form than the AL and AD.



Streamwise velocity contours in cross-section 1D downstream.



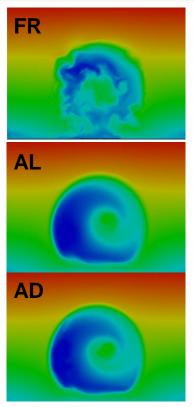
Mean streamwise velocity 1D downstream for various azimuth positions



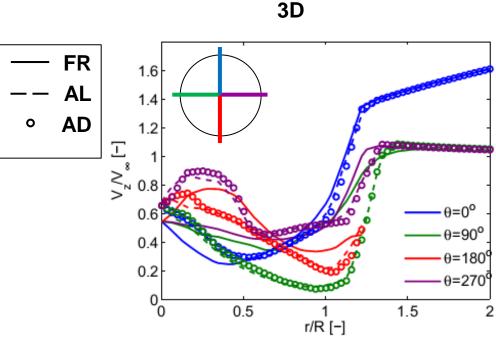
Test cases

- Sheared inflow
- Yawed inflow

- Good agreement in predicted near wake deficit
- AL and AD in close agreement
- ➤ Wake of FR develops faster into a bell shaped form than the AL and AD.



Streamwise velocity contours in cross-section 3D downstream.

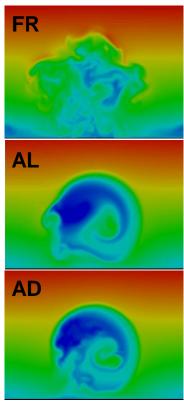


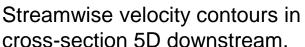
Mean streamwise velocity 3D downstream for various azimuth positions

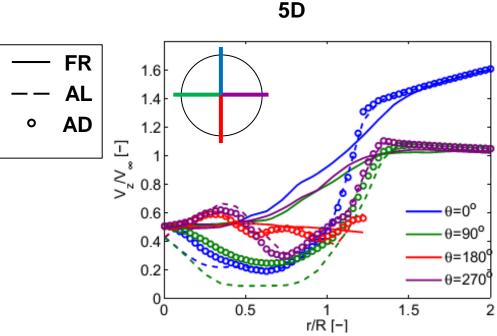


- Sheared inflow
- Yawed inflow

- Good agreement in predicted near wake deficit
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- ➤ Wake of FR develops faster into a bell shaped form than the AL and AD.







Mean streamwise velocity 5D downstream for various azimuth positions



- Sheared inflow
- Yawed inflow

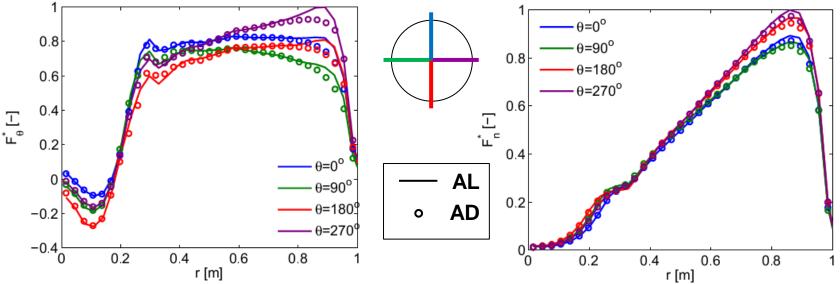
- $V_{\infty} = 8 \text{ m/s}$
- > Yaw error of 20°



Test cases

- Sheared inflow
- Yawed inflow

Load predictions in good agreement



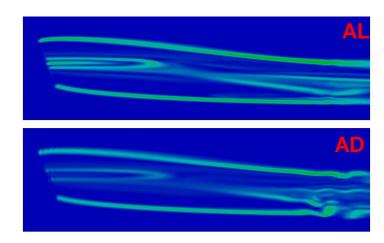
Spanwise distribution of normal and tangential loads at various azimuth positions



Test cases

- Sheared inflow
- Yawed inflow

Similar vorticity contours



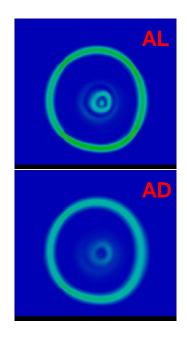
Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center. 30



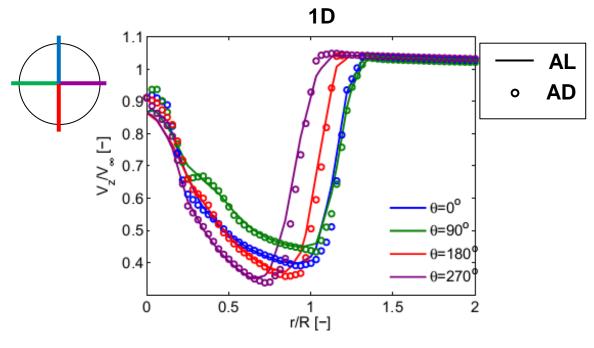
Test cases

- Sheared inflow
- Yawed inflow

Good agreement in predicted wake deficit and wake structure



Vorticity magnitude contours in cross-section 1D downstream.



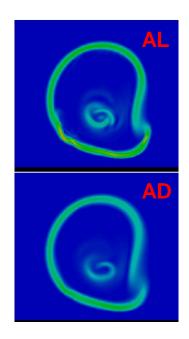
Mean streamwise velocity 1D downstream for various azimuth positions



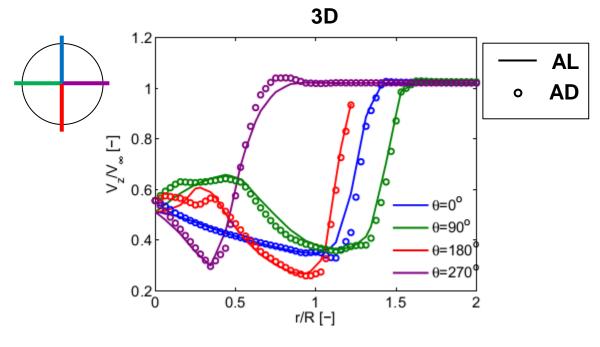
Test cases

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Good agreement in predicted wake deficit and wake structure



Vorticity magnitude contours in cross-section 3D downstream.



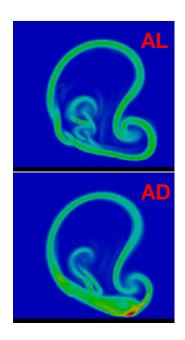
Mean streamwise velocity 3D downstream for various azimuth positions



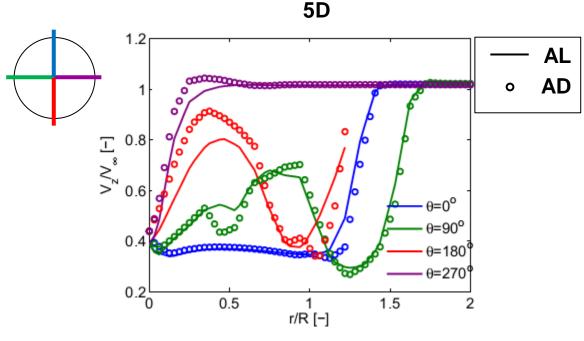
Test cases

- Sheared inflow
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Good agreement in predicted wake deficit and wake structure



Vorticity magnitude contours in cross-section 5D downstream.



Mean streamwise velocity 5D downstream for various azimuth positions

Conclusions



Sheared inflow

- ➤ Three models show good agreement in axial velocity up to 2D downstream of the turbine.
- ➤ Further downstream the FR simulation predicts a faster smearing of the mean gradients
- Much higher turbulence in the FR simulation
- Generally good agreement between AL and AD for all downstream position.

Yawed inflow

- Good resemblance between wake behavior predicted using AL and AD.
- ➤ AD representation as accurate as AL even in nonuniform inflow.