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Vibration tolerance of micro-electrodes

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Abstract.

Micro four-point probe measurements can be used for extremely accurate electrical characterization of advanced electronic materials. The measurements are, however, sensitive to environmental vibrations which may compromise measurement quality severely. We have experimentally measured the vibration tolerance of two different micro-electrode designs (straight and three-way flexible) on two different samples (Indium-Tin-Oxide and highly doped Silicon). We show that three-way flexible electrodes are significantly more vibration tolerant (at least 50 times) compared to their straight counterparts. We have theoretically analyzed the vibration tolerance of the electrode designs and show excellent agreement between theory and experiment; as a result the theoretical framework may be used to predict vibration tolerance of any micro-electrode.

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1. Introduction

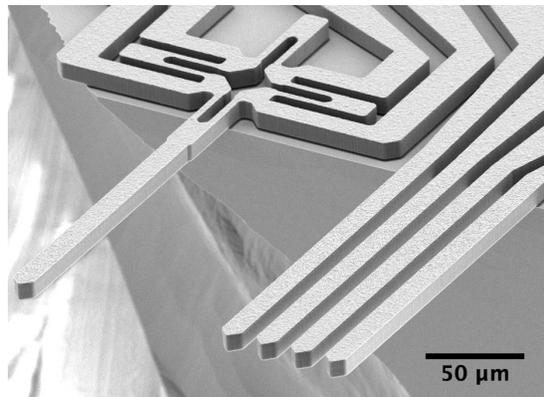
For more than a century the four-point probe (4PP) has been a preferred method for precise electrical characterization of materials [1–3]. However, the size of the probes and the contact force during engagement have limited its use for measurements on ultra thin materials [4, 5].

Miniaturization obtained by use of microfabrication technologies has had a strong impact on micromanipulator performance [6] and also improved the performance of 4PP with the introduction of the micro four-point probe (M4PP) as an alternative to conventional 4PP [7]. During the past decade, M4PP measurements have become the most reliable method for electrical characterization of ultra-thin semiconducting layers [8]. Today, fully automatic systems can perform sheet resistance [9] and micro Hall effect [10, 11] measurements in scribe-line test pads using lock-in technique and configuration switching combined with advanced data treatment algorithms [12] for state-of-the-art reliability [4, 5, 13].

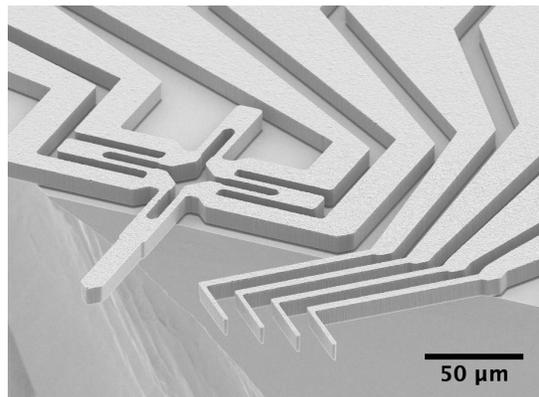
An M4PP typically comprises (at least) four cantilever electrodes extending from a silicon probe body. The cantilever electrodes are silicon or silicon dioxide beams coated with metal (e.g., Ni). During measurements the contact force of each electrode is around 10-100 μN , i.e., around five orders of magnitude lower than that of a conventional 4PP [8]. M4PPs can be made of metal coated polymer [14, 15], or alternatively measurements may be done with independently movable probes [16].

M4PP measurements rely on a static contact between the micro-electrodes and the test sample during the full duration of the data acquisition which lasts for several seconds. However, fully automatic metrology systems are most often placed in production environments with significant ambient vibrational noise, which could result in unintended motion of the contact points. If a static contact is not maintained during a measurement sequence, the result is discarded due to stringent measurement requirements imposed by the need for high reproducibility [12, 17–20].

The original M4PP electrodes were simple straight cantilevers (I-beams) [7] as shown in Figure 1a. Later, high aspect ratio, three-way flexible cantilever electrodes (L-beams, Figure 1b) were proposed for reducing abrasive wear of the electrode tips by forming a static contact [21, 22].



(a)



(b)

Figure 1: SEM images of micro four-point probes. (a) I-beam probes with straight electrodes. (b) L-beam probes with more flexible electrode beams. The images also show Wheatstone bridge strain-gauges (for surface detection) on both probes.

Here we experimentally determine the vibration tolerance of I-beam and L-beam cantilevers on two different sample surfaces as a function of engagement depth (i.e., contact force). We show experimentally that the L-beam has a higher vibration tolerance than that of the traditional I-beam. We also develop a theoretical model for the vibration tolerance of any cantilever with a known compliance tensor. We continue to calculate the vibration tolerance of the I-beam and L-beam cantilevers, and show that the model predictions are in excellent agreement with the experimental results.

2. Theory

In a typical measurement instrument, the electrodes are engaged with the sample at a tilt angle ϕ (e.g., $\phi = 30^\circ$) and an engagement depth δ_z (e.g., $\delta_z = 500$ nm) to provide sufficient contact force F_N for reliable electrical contact. The electrode-sample geometry is illustrated in Figure 2a for both I-beam and L-beam electrodes. Figure 2a also illustrates the sample based Cartesian coordinate system (x, y, z) we shall use for analysis of vibration tolerance, while Figure 2b shows the cantilever based coordinate system (x'', y'', z'') which we shall use for analysis of the elastic behavior of the beams. Figure 2b also defines the geometry of the L-beam, which is attached to the probe body at the hinge angle θ as illustrated in Figure 2a. The analysis of vibration tolerance is simplified significantly if the electrode-sample contact is assumed to be point like such that the contact point can support forces, but not torques. Then the relation between the deflection vector δ and force \mathbf{F} on the electrode-tip is characterized by a stiffness $[\mathbf{K}]$ or compliance tensor $[\mathbf{C}] = [\mathbf{K}]^{-1}$, i.e., $\mathbf{F} = [\mathbf{K}]\delta$ and $\delta = [\mathbf{C}]\mathbf{F}$ as discussed by F. Wang et al. [22].

During electrode-sample engage two extreme cases may occur: Either static contact is obtained, i.e., $\delta = \delta_z \hat{\mathbf{z}}$ where $\hat{\mathbf{z}}$ is the unit vector normal to the surface of the sample. Or the cantilever slides on the surface until forces tangential to the surface vanish, i.e., $\mathbf{F} = F_N \hat{\mathbf{z}} = C_{zz}^{-1} \delta_z \hat{\mathbf{z}}$. In the analysis we shall assume sliding contact. During measurements, when the sample and probe are in contact, the sample may move relative to the probe body due to environmental noise vibrations, and then the contact point may or may not move on the sample depending on the vibration amplitude and properties of the contact and the electrode. We shall assume that the contact is maintained if the magnitude of the tangential force F_T is less than the total normal force $F_{N,\text{tot}}$ times the friction coefficient μ , i.e., $|F_T| \leq \mu F_{N,\text{tot}}$, otherwise the contact point will slide on the surface and compromise measurement quality.

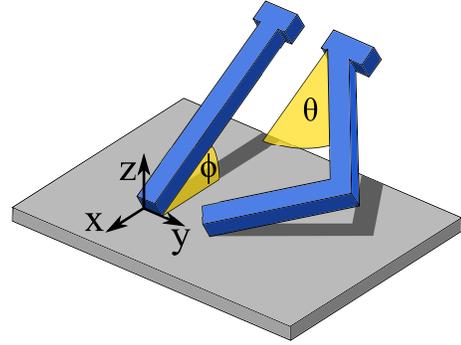
Consider now in-plane sample-body vibrations $\tilde{\delta}(t)$ with the amplitude A at an arbitrary polar angle ψ to the x -axis

$$\tilde{\delta}(t) = A \sin(\omega t) \begin{bmatrix} \cos \psi \\ \sin \psi \\ 0 \end{bmatrix}, \quad (1)$$

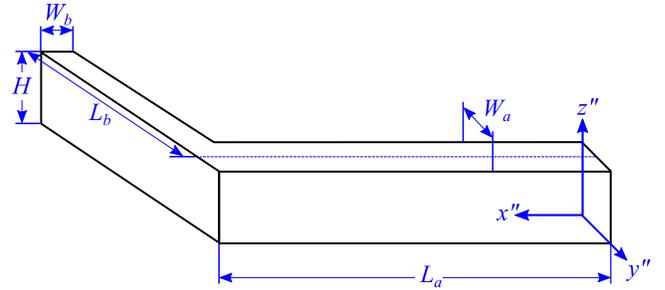
where ω is the angular vibration frequency and t is time. The resulting force due to engagement with sliding contact and added vibration is $\mathbf{F} = F_N \hat{\mathbf{z}} + [\mathbf{K}]\tilde{\delta}(t)$ and thus the total normal force $F_{N,\text{tot}}$ and tangential force F_T become

$$F_{N,\text{tot}} = C_{zz}^{-1} \delta_z + K_N(\psi) A \sin(\omega t), \quad (2)$$

$$F_T = K_T(\psi) A \sin(\omega t), \quad (3)$$



(a)



(b)

Figure 2: (a) Sketch of the electrode-sample geometry in the sample coordinate system. Both the I-beam and the L-beam electrodes are engaged with the sample at the tilt angle ϕ . The L-beam is attached to the probe body with the hinge angle θ . (b) Sketch of the L-beam electrode in the cantilever coordinate system illustrating the definition of geometry symbols.

where $K_N(\psi)$ and $K_T(\psi)$ are effective normal and transverse spring constants, respectively. In terms of the stiffness tensor elements the spring constants are

$$K_N(\psi) = K_{zx} \cos \psi + K_{zy} \sin \psi, \quad (4)$$

$$K_T(\psi) = \quad (5)$$

$$\sqrt{(K_{xx} \cos \psi + K_{xy} \sin \psi)^2 + (K_{yx} \cos \psi + K_{yy} \sin \psi)^2}.$$

The worst case scenario occurs when $|\sin(\omega t)| = 1$, and since $K_T(\psi)$ is always positive the vibration amplitude must fulfill

$$\frac{A}{\delta_z} \leq \frac{\mu C_{zz}^{-1}}{K_T(\psi) + \mu |K_N(\psi)|} \equiv \Gamma(\psi) \quad (6)$$

to avoid a sliding contact point according to the criterion $|F_T| \leq \mu F_{N,\text{tot}}$ discussed above. The vibration tolerance, in units of the engagement depth, for a vibration in the direction ψ is thus $\Gamma(\psi)$.

Depending on the design of the electrodes two simplified cases can be identified, i.e., in case $K_T(\psi) \gg \mu |K_N(\psi)|$ the vibration tolerance $\Gamma(\psi) \simeq \mu / [C_{zz} K_T(\psi)]$ is proportional to the

friction coefficient, and in case $K_T(\psi) \ll \mu |K_N(\psi)|$ the vibration tolerance $\Gamma(\psi) \simeq 1/[C_{zz} |K_N(\psi)|]$ is independent on the friction coefficient.

The vibration tolerance for the I-beam is minimum at polar angles $\psi = 0 \pm \pi$ and maximum at $\psi = \pm \frac{\pi}{2}$, and the same is roughly the case for the L-beam, albeit with a negligible offset. At the minimum and maximum the vibration tolerance becomes

$$\Gamma(0) = \frac{\mu C_{zz}^{-1}}{\sqrt{K_{xx}^2 + K_{yy}^2 + \mu |K_{zx}|}}, \quad (7)$$

$$\Gamma\left(\frac{\pi}{2}\right) = \frac{\mu C_{zz}^{-1}}{\sqrt{K_{xy}^2 + K_{yy}^2 + \mu |K_{zy}|}}, \quad (8)$$

respectively.

2.1. Compliance tensors

To proceed further, the compliance tensors of the cantilevers must be calculated and here we use Euler's beam equation [23] and the double-primed cantilever coordinate system of Figure 2b.

The calculation of the compliance tensor describing the I-beam, which is a prismatic beam of length L_I , thickness H_I and width W_I , proceeds by assuming that one end is rigidly clamped to the probe body. The resulting I-beam compliance tensor $[\mathbf{C}_I'']$ in the cantilever coordinate system is

$$[\mathbf{C}_I''] = \begin{bmatrix} \frac{L_I}{EW_I H_I} & 0 & 0 \\ 0 & \frac{4L_I^3}{EW_I^3 H_I} & 0 \\ 0 & 0 & \frac{4L_I^3}{EW_I H_I^3} \end{bmatrix} \quad (9)$$

where $E = 170$ GPa is Young's modulus of the beam material. Note, the element $C_{I,xx}''$ is a low compliance element which is controlled by longitudinal deformation. In the remaining non-zero elements the effect of longitudinal deformation is insignificant and thus ignored.

The L-beam consists of two prismatic beams, the first beam (length L_a , width W_a , and height H) is clamped to the probe body, while the second beam (length L_b , width W_b , and height H) is connected to the first beam at a 90° degree elbow. The compliance tensor $[\mathbf{C}_L'']$ for the L-Beam cantilever was calculated by Wang et al. [22]

$$[\mathbf{C}_L''] = \frac{1}{E} \begin{bmatrix} \frac{12L_a L_b^2}{W_a^3 H} + \frac{4L_b^3}{W_b^3 H} & -\frac{6L_a^2 L_b}{W_a^3 H} & 0 \\ -\frac{6L_a L_b}{W_a^3 H} & \frac{4L_a^3}{W_a^3 H} & 0 \\ 0 & 0 & \frac{4L_a^3}{W_a H^3} + \frac{4L_b^3}{W_b H^3} + \frac{EL_b^2}{k_\tau} \end{bmatrix}, \quad (10)$$

where k_τ is the torsion constant of the first prismatic bar of the L-beam [24]

$$k_\tau = \frac{EW_a^3 H}{2(1+\nu)L_a} \left(\frac{1}{3} - \frac{64W_a}{\pi^5 H} \beta \right), \quad (11)$$

L-beam		I-beam	
Symbol	Value (μm)	Symbol	Value (μm)
H	9.7	H_I	9.9
L_a	92.1	L_I	201.0
L_b	36.9		
W_a	4.8	W_I	12.0
W_b	2.3		
θ	45°	θ_I	0°
ϕ	30°	ϕ_I	30°

Table 1: Dimensions of the two cantilever designs used as defined in Figure 2. The electrode pitch of both probes is 20 μm .

where ν is Poisson's ratio and

$$\beta = \sum_{n=1}^{\infty} \frac{1}{(2n-1)^5} \tanh \frac{(2n-1)\pi H}{2W_a}. \quad (12)$$

These compliance tensors describe the deflection of the cantilever tips relative to the probe body in response to an applied force. However, the probe body may not be perfectly rigidly attached to the bulk of the instrument, and thus it is necessary to include also a system compliance $[\mathbf{C}_\varepsilon'']$ to fully describe the electrode-sample interaction. Since the impact of system compliance is significant only in low compliance directions of the cantilevers a system compliance tensor with only a single non-zero element is sufficient, i.e.,

$$[\mathbf{C}_\varepsilon''] = \begin{bmatrix} \varepsilon & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \quad (13)$$

with the non-zero element assigned the value $\varepsilon = 0.2$ mm/N, which is an insignificant compliance compared to all compliance elements except $C_{I,xx}''$, which is an order of magnitude smaller than ε . As a result the total compliance tensors are $[\mathbf{C}_{I_\varepsilon}'] = [\mathbf{C}_I''] + [\mathbf{C}_\varepsilon'']$ and $[\mathbf{C}_{L_\varepsilon}'] \simeq [\mathbf{C}_L'']$ for the I-beam and L-beam, respectively. The method used for determining the magnitude of ε is described in section 4.

For vibration tolerance calculations in the sample coordinate-system the compliance tensors are then rotated using the rotation matrix [22]

$$[\mathbf{T}] = \begin{bmatrix} \cos \theta \cos \phi & -\sin \theta & -\cos \theta \sin \phi \\ \sin \theta \cos \phi & \cos \theta & -\sin \theta \sin \phi \\ \sin \phi & 0 & \cos \phi \end{bmatrix}. \quad (14)$$

Table 1 shows the dimensions of the I-beam and L-beam cantilevers used in vibration tolerance calculations. The dimensions reported in Table 1 were measured on SEM images of the beams. For the calculations the elastic parameters, Young's modulus

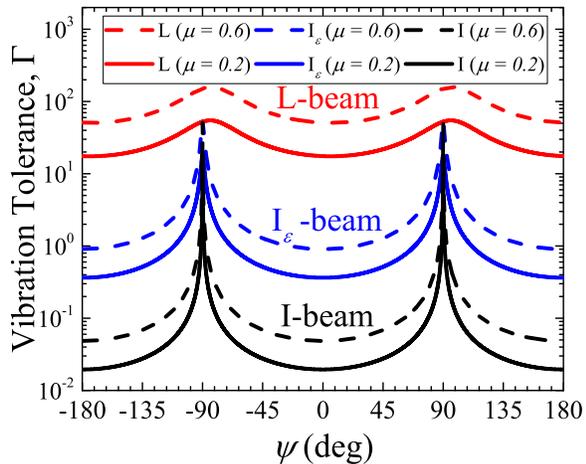


Figure 3: Calculated normalized vibration tolerance Γ for I-beam (black) and L-beam (red) cantilevers as a function of the polar angle ψ assuming friction coefficients $\mu = 0.2$ (full) and 0.6 (dashed). The curves in blue, labeled I_ϵ -beam, are calculated by taking into account the system compliance.

$E = 170$ GPa and Poisson's ratio $\nu = 0.22$, were used [25].

Figure 3 shows the calculated vibration tolerances for I-beam (black) and L-beam (red) electrodes as a function of the polar angle ψ for two different values ($\mu = 0.2$ full lines and $\mu = 0.6$ dashed lines) of the electrode-sample friction coefficient. The curves labelled I_ϵ -beam (blue) results when the system compliance is included; the system compliance has a major effect on the calculated vibration tolerance for the I-beam as seen. The calculated vibration tolerance of the L-beam is roughly a factor of 50 higher than that of the I_ϵ -beam.

3. Experiment

The vibration tolerance of the cantilever probes (I_ϵ -beam and L-beam) shown in Figure 1 was characterized experimentally by intentional vibration of the sample during dual configuration sheet resistance measurements. The sample to be measured was placed on a calibrated piezoelectric element, which was driven at a frequency of 100 Hz and with the vibration amplitude A steadily increased during a measurement series. The direction of the sample vibration was aligned to the direction of minimum vibration tolerance for the cantilevers, i.e., $\psi \simeq 0$. Measurements were done on two different samples, i.e., on a silicon sample (Si) with a highly doped p-type shallow junction and on an indium-tin-oxide (ITO) sample with sheet resis-

tances of 380 Ω and 90 Ω , respectively. Sheet resistance measurements were done using an M200 tool from CAPRES A/S, and performed according to industry standards with a 1% median filter on the dual configuration sheet resistance measurements [26].

Measurement failure is interpreted as caused by motion of the cantilever contact points on the sample surface. Thus, the absolute vibration tolerance $\Gamma\delta_z$ was experimentally defined as the minimum vibration amplitude A where the dual configuration sheet resistance measurement failed five times in a row (Note in clarification: at larger amplitudes resistance measurements almost always fail, while smaller amplitudes result in less frequent measurement failures.). In this fashion the vibration tolerance was measured as a function of the engagement depth δ_z , which was varied in the range from 500 to 3000 nm, with an estimated error of 200 nm, as controlled by the probe stage on the M200. The engagement depth is measured as the set travel of the probe stage beyond detection of the surface. Representative measurements of the vibration tolerances of the I_ϵ -beams and the L-beams on the two samples are reported in Figure 4.

4. Results and discussion

The experimental vibration tolerance data in Figure 4 show that, indeed, the absolute vibration tolerance increases approximately linearly with increased engagement depth δ_z for each combination of probe (I-beam or L-beam) and sample (Si or ITO) as expected. The vibration tolerance of the L-beam is much larger than that of the I_ϵ -beam, and is, within measurement error, independent of the properties of the sample. This behavior is in stark contrast to the vibration tolerance of the I_ϵ -beam, which is strongly dependent of the properties of the sample, i.e., the vibration tolerance on Si is an order of magnitude lower than that on ITO. This sample dependence of the vibration tolerance can be understood by inspecting post-measurement SEM images (not shown) of the sample surfaces, which show rather deep indentations from the I_ϵ -beam on the ITO sample, but not on the Si sample. The indentations on the ITO sample effectively prevents the normal sliding motion on the surface; thereby the vibration tolerance is increased significantly.

In Figure 4 error bars are not reported, but there are sources of error on both axes. The larger error is that of the engagement depth, which is determined based on a threshold signal from the built-in strain gauge on the probes (Figure 1) and the motion of the calibrated probe stage of the M200 tool beyond this threshold. The threshold detection is expected to give a systematic error of at most 200 nm. The probe stage has a resolution of 2.5 nm and a position stability of

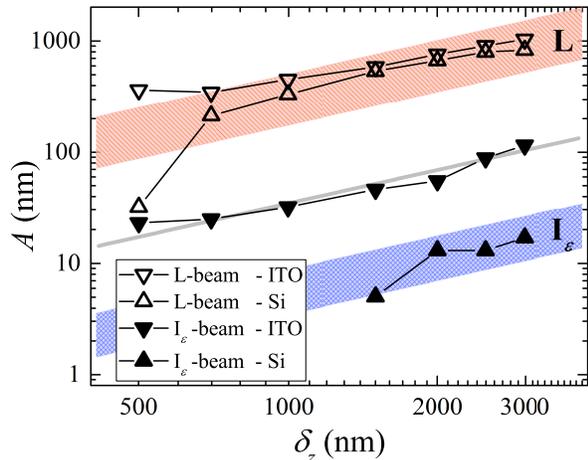


Figure 4: Experimental absolute vibration tolerance A as function of engagement depth δ_z for I_ϵ -beam and L-beam electrodes on Si and ITO samples. Theoretical predictions of the vibration amplitude for I_ϵ -beam and L-beam electrodes are shown as the blue and red bands respectively, for friction coefficients in the range $\mu = 0.2$ to 0.6 . The full grey line is the theoretical prediction for the I_ϵ -beam with an artificially large friction coefficient ($\mu \rightarrow \infty$), which agrees well with I_ϵ -beam measurements on ITO.

10 nm peak-to-peak; thus it contributes with a random error of at most 5.6 nm. The vibration amplitude is much more accurately determined, and here the main source of error is environmental vibration noise (a few nm) during the measurements. This noise affected the measurements for the I_ϵ -beam on Si where data could not be obtained at low engagement depths (below 1500 nm).

The experimental vibration tolerance of the L-beam is in very good agreement with the calculated vibration tolerance as illustrated by the red band (top) in Figure 4, which shows calculated vibration tolerance for friction coefficients in the range from $\mu = 0.2$ to 0.6 ; i.e., almost all measurement points for both samples are within this band. The blue band (bottom) in Figure 4 illustrates a similar calculation for the I_ϵ -beam assuming a system compliance $\epsilon = 0.2$ mm/N, and again the agreement between experiment and theory is very good and essentially all measurement points on Si are within the band defined by the friction coefficient range (0.2 to 0.6).

The vibration tolerance of the I_ϵ -beam on ITO is an order of magnitude higher than that on Si and the theoretical prediction (blue band in Figure

4), but as mentioned above the indentation of the electrode in the sample prevents the usual sliding motion on the surface. We may model this behavior by allowing an artificially high friction coefficient (i.e., $\mu \rightarrow \infty$) and then the normalized vibration tolerance becomes $\Gamma(0) \simeq 1/(C_{zz}|K_{zx}|)$, which is independent on μ . By fitting this simplified expression to the measurement data for the I_ϵ -beam on ITO using the system compliance ϵ as the fitting parameter produces the full grey line in Figure 4 for $\epsilon = 0.2$ mm/N, and the agreement with measurements is excellent. Thus, this experiment essentially measures the system compliance, and the resulting system compliance was also used to explain the data for the I_ϵ -beam on Si experiments in the previous paragraph.

5. Conclusion

We have measured the vibration tolerance of L-beam and I_ϵ -beam electrodes on ITO and Si samples. The measurements show that L-beam electrodes are at least 50 times more vibration tolerant than I_ϵ -beam electrodes. We have explained this behavior theoretically and the model of vibration tolerance we have developed is in excellent agreement with the experiments. Our theoretical framework may thus serve as a valuable tool for future micro-electrode design focusing on improved vibration tolerance.

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