



An improved tensorial implementation of the incremental harmonic balance method for frequency-domain stability analysis

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- ▶ Applications in dynamic systems
 - ▶ SDOF and MDOF nonlinear dynamic systems
 - ▶ Finite element models of nonlinear mechanical systems
- ▶ Analyses
 - ▶ Time-periodic response analysis
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The governing equation of finite element model of nonlinear mechanical system with geometric nonlinearity can be written as

$$M \frac{d^2 \mathbf{u}}{dt^2} + C \frac{d\mathbf{u}}{dt} + \mathbf{g}(\mathbf{u}) = \mathbf{f}(\omega t) \quad (1)$$

M : the mass matrix

C : the linear damping matrix

\mathbf{g} : the nonlinear internal force vector (e.g. quadratic/cubic polynomial function)

\mathbf{f} : the time-periodic external force vector

ω : the circular frequency $\omega = \frac{2\pi}{T}$, where T is the time period



- ▶ Step 1: Introduce a new time scale τ

$$\tau = \omega t = 2\pi(t/T) \quad (2)$$

We have

$$\frac{du}{dt} = \omega \frac{du}{d\tau}, \quad \frac{d^2u}{dt^2} = \omega^2 \frac{d^2u}{d\tau^2} \quad (3)$$

Equation (1) can be written as

$$\omega^2 M \frac{d^2u}{d\tau^2} + \omega C \frac{du}{d\tau} + g(u) = f(\tau) \quad (4)$$

- ▶ Step 2: Fourier expansion of time-periodic response

$$\mathbf{u}(\tau) = \mathbf{s}(\tau) \mathbf{q} \quad (5)$$

where $\mathbf{s}(\tau)$ is Fourier basis (i.e. cosine, sine), \mathbf{q} is a vector of Fourier coefficients. Substituting equation (5) into equation (4) and applying the Galerkin method, *the governing equation in the frequency domain is*

$$(\omega^2 \bar{\mathbf{M}} + \omega \bar{\mathbf{C}}) \mathbf{q} + \bar{\mathbf{g}}(\mathbf{q}) = \bar{\mathbf{f}} \quad (6)$$

The governing equation can also be written in a compact form as

$$\mathbf{r}(\mathbf{q}, \omega) = 0 \quad (7)$$

where $\mathbf{r}(\mathbf{q}, \omega) = (\omega^2 \bar{\mathbf{M}} + \omega \bar{\mathbf{C}}) \mathbf{q} + \bar{\mathbf{g}}(\mathbf{q}) - \bar{\mathbf{f}}$.

- ▶ Step 3: Consider an unknown solution in the neighborhood of a given solution expressed by q_0 and ω_0

$$q = q_0 + \Delta q, \quad \omega = \omega_0 + \Delta \omega. \quad (8)$$

Substituting equation (8) into equation (7), we have

$$r(q_0 + \Delta q, \omega_0 + \Delta \omega) = 0 \quad (9)$$

The incremental form of the frequency-domain governing equation is

$$K_q \Delta q + K_\omega \Delta \omega = -r(q_0, \omega_0) \quad (10)$$

$$K_q = \left. \frac{\partial r(q, \omega)}{\partial q} \right|_{q=q_0, \omega=\omega_0}, \quad K_\omega = \left. \frac{\partial r(q, \omega)}{\partial \omega} \right|_{q=q_0, \omega=\omega_0}. \quad (11)$$



- ▶ The Jacobian matrices K_q and K_w are explicitly expressed as

$$K_q = \omega_0^2 \bar{M} + \omega_0 \bar{C} + \bar{K}_t(q_0), \quad (12)$$

$$K_w = 2\omega_0 \bar{M} + \bar{C}. \quad (13)$$

where

$$\bar{K}_t(q_0) = \left. \frac{\partial \bar{g}(q)}{\partial q} \right|_{q=q_0} \quad (14)$$

- ▶ Consider a disturbed response at u_0 with a small disturbance δ

$$u = u_0 + \delta \quad (15)$$

Substituting equation (15) into equation (4) and retaining the first-order approximation, the following equation is obtained as

$$\omega_0^2 M \frac{d^2 \delta}{d\tau^2} + \omega_0 C \frac{d\delta}{d\tau} + K_t(q_0) \delta = 0 \quad (16)$$

- ▶ Express the disturbance δ as

$$\delta = e^{\lambda\tau} s p \quad (17)$$

where λ and p denote the Floquet exponent and vector, respectively.

- ▶ The first- and second-order derivatives of δ are

$$\frac{d\delta}{d\tau} = \lambda e^{\lambda\tau} s p + e^{\lambda\tau} \frac{ds}{d\tau} p \quad (18)$$

$$\frac{d^2\delta}{d\tau^2} = \lambda^2 e^{\lambda\tau} s p + 2\lambda e^{\lambda\tau} \frac{ds}{d\tau} p + e^{\lambda\tau} \frac{d^2s}{d\tau^2} p \quad (19)$$

- ▶ Substituting equations (17–19) into equation (16) and applying the Galerkin method, a quadratic eigenvalue problem is obtained as

$$(J_2\lambda^2 + J_1\lambda + J_0) p = 0 \quad (20)$$

$$J_0 = K_q(q_0, \omega_0), \quad J_1 = 2\omega_0^2 M \otimes h^{(1)} + \omega_0 C \otimes h^{(2)}, \quad J_2 = \omega_0^2 M \otimes h^{(2)} \quad (21)$$

The following frequency-domain quantities are computed as

$$\begin{aligned}
 \bar{\mathbf{M}} &= \mathbf{M} \otimes \mathbf{h}^{(0)}, & \bar{\mathbf{C}} &= \mathbf{C} \otimes \mathbf{h}^{(1)}, \\
 \bar{\mathbf{g}} &= \begin{bmatrix} \mathbf{h}^{(2)} \mathcal{G}_1 \\ \vdots \\ \mathbf{h}^{(2)} \mathcal{G}_{N_{dof}} \end{bmatrix}, & \bar{\mathbf{f}} &= \begin{bmatrix} \mathbf{h}^{(2)} \mathcal{F}_1 \\ \vdots \\ \mathbf{h}^{(2)} \mathcal{F}_{N_{dof}} \end{bmatrix}
 \end{aligned} \tag{22}$$

where \otimes denotes the Kronecker operator, \mathcal{G}_i and \mathcal{F}_i denote the Fourier coefficients of the internal and external forces for the i^{th} degree of freedom.

The key matrix \bar{K}_t is obtained with the assistance of a three-dimensional tensor $h^{(3)}$ and the operation of tensor contraction as

$$\bar{K}_t = \begin{bmatrix} h^{(3)} : \mathcal{K}_{1,1} & \dots & h^{(3)} : \mathcal{K}_{1,N_{dof}} \\ \vdots & \ddots & \vdots \\ h^{(3)} : \mathcal{K}_{N_{dof},1} & \dots & h^{(3)} : \mathcal{K}_{N_{dof},N_{dof}} \end{bmatrix} \quad (23)$$

where the symbol “:” denotes the tensor contraction operation, and $\mathcal{K}_{i,j}$ denotes a vector of Fourier coefficients corresponding to the component at the position (i, j) of $K_t(u(\tau))$.

$\mathbf{h}^{(0)}$, $\mathbf{h}^{(1)}$, and $\mathbf{h}^{(2)}$ are two-dimensional matrices given as

$$\mathbf{h}^{(0)} = -\frac{1}{2} \begin{bmatrix} 0 & 0 & 0 \\ 0 & \hat{\mathbf{I}} & 0 \\ 0 & 0 & \hat{\mathbf{I}} \end{bmatrix}, \quad \hat{\mathbf{I}} = \begin{bmatrix} 1 & & \\ & \ddots & \\ & & N_H^2 \end{bmatrix}_{N_H \times N_H} \quad (24)$$

$$\mathbf{h}^{(1)} = \frac{1}{2} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & \bar{\mathbf{I}} \\ 0 & -\bar{\mathbf{I}} & 0 \end{bmatrix}, \quad \bar{\mathbf{I}} = \begin{bmatrix} 1 & & \\ & \ddots & \\ & & N_H \end{bmatrix}_{N_H \times N_H} \quad (25)$$

$$\mathbf{h}^{(2)} = \frac{1}{2} \begin{bmatrix} 2 & 0 & 0 \\ 0 & \mathbf{I} & 0 \\ 0 & 0 & \mathbf{I} \end{bmatrix}, \quad \mathbf{I} = \begin{bmatrix} 1 & & \\ & \ddots & \\ & & 1 \end{bmatrix}_{N_H \times N_H} \quad (26)$$

$\mathbf{h}^{(3)}$ is a three-dimensional tensor defined as

$$\mathbf{h}^{(3)}(i, j, k) = \frac{1}{2\pi} \int_0^{2\pi} \mathcal{S}_i(\tau) \mathcal{S}_j(\tau) \mathcal{S}_k(\tau) d\tau \quad (27)$$

where $i, j, k = 1, \dots, 2N_H + 1$, and $\mathcal{S} = [1 \ \cos(\tau) \ \dots \ \cos(N_H\tau) \ \sin(\tau) \ \dots \ \sin(N_H\tau)]$.

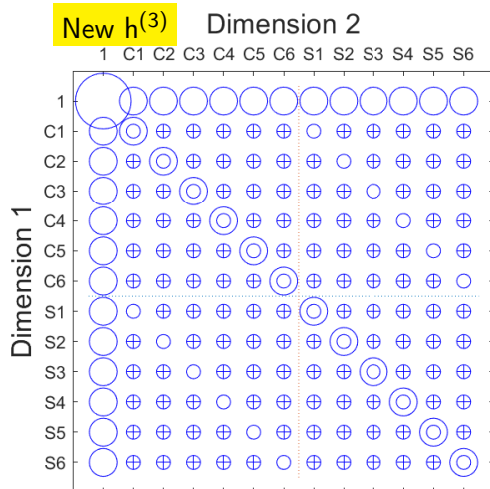
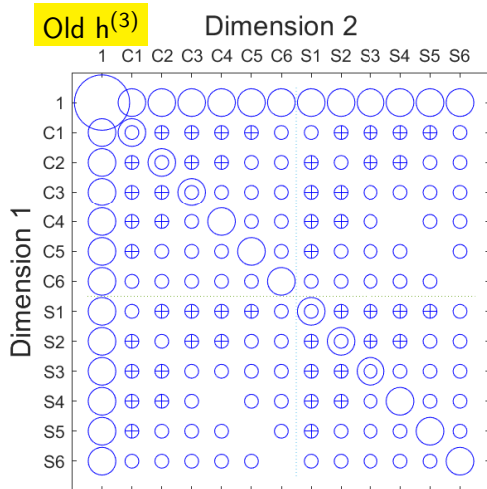
An improved version of the three-dimensional tensor $\mathbf{h}^{(3)}$ is given as

$$\mathbf{h}^{(3)}(i, j, k) = \frac{1}{2\pi} \int_0^{2\pi} \mathcal{S}_i(\tau) \mathcal{S}_j(\tau) \widehat{\mathcal{S}}_k(\tau) d\tau \quad (28)$$

where $i, j = 1, \dots, 2N_H + 1$, $k = 1, \dots, 4N_H + 1$, and $\widehat{\mathcal{S}}$ is given as

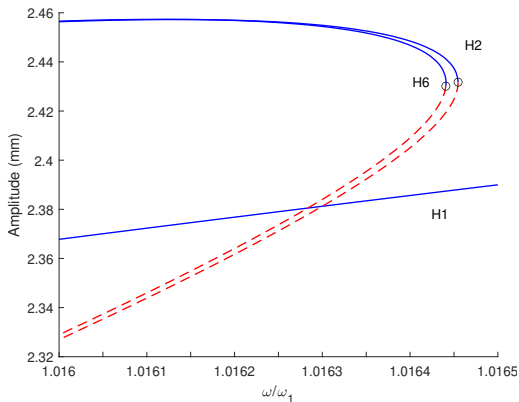
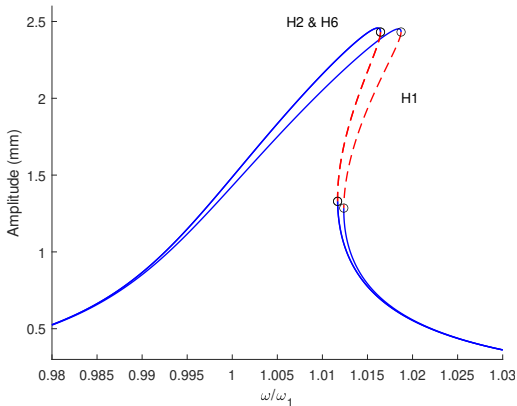
$$\widehat{\mathcal{S}} = [1 \ \cos(\tau) \ \dots \ \cos(2N_H\tau) \ \sin(\tau) \ \dots \ \sin(2N_H\tau)]$$

Visualization of the two versions of the three-dimensional tensor $h^{(3)}$



- ▶ Finite element model of a clamped-clamped beam
Young's modulus $E = 205$ Gpa, mass density $\rho = 7800$ Kg/m³
cross section $0.01\text{m} \times 0.01\text{m}$, length $L = 150\sqrt{I/A}$
 $C = \alpha M$, where $\alpha = 15.58$, damping ratio 1% for 1st flexural mode
20 Euler-Bernoulli beam element
Geometric nonlinearity from mid-plane stretching effect
- ▶ For the old version of $h^{(3)}$ and $N_H = 1$, response analysis fails around the resonance peak
- ▶ For the new version of $h^{(3)}$ and $N_H = 1$, response analysis and stability analysis are performed smoothly.

Results for response and stability with the new version of $h^{(3)}$ and $N_H = 1, 2, 6$



- ▶ Tensorial implementation of the incremental harmonic balance method
- ▶ An improved version of the three-dimensional tensor $h^{(3)}$
 - ▶ Accurate computation of Jacobian matrix up to cubic order
- ▶ Results for response and stability analyses can be obtained with a small value of N_H
- ▶ Future work
 - ▶ applications in sensitivity analysis and gradient based structural optimization
 - ▶ further reduction of computational cost of frequency-domain stability analysis